



**PERFORMANCE COMPARISON OF FRACTIONAL ORDER PID CONTROLLER  
AND LINEAR QUADRATIC REGULATOR FOR LEVEL CONTROL IN COUPLED  
TANK SYSTEMS**

**M.SC. THESIS**

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**A THESIS SUBMITTED THE  
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## DECLARATION

I hereby declare that this thesis work is my original work, has not been presented for a degree in this or any other university, and all sources of materials used for the thesis work have been fully acknowledged.

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LIST OF SYMBOLS

$\lambda$ .....	Fractional order lamda
$\mu$ .....	Fractional order miu
p.....	Proportional
I.....	Integral
D.....	Derivative
$k_p$ .....	Proportional gain
$k_I$ .....	Integral gain
$k_d$ .....	Derivative gain
k.....	A gain of second-order plant
e(t).....	Error signal
r(t).....	Input
G(s).....	Controller transfer function(s-domain)
$G_p(s)$ .....	The transfer function of the plant
S.....	Shaped or oscillatory open-loop dynamics
m.....	Relative dominance
$G_a(j\omega)$ .....	The transfer function of the frequency domain
$T_p$ .....	peak time
$T_r$ .....	Rise time
$T_s$ .....	Settling time
$\xi$ .....	damping ratio of the second-order plant
$\omega_n$ .....	The natural frequency of the second-order plant
$Q_{in}(t)$ .....	The flow of liquid into tanks
$Q_{out}(t)$ .....	Flow of liquid
Im.....	Imaginary
Re.....	Real

- Q.....Weighting matrix
- R.....Scaler quantity
- K.....State feedback gain
- A.....(nxn) square state matrix
- B.....(nxr) input matrix
- C.....(mrx) direct feedback matrix
- P.....Riccati matrix
- $h(t)$ .....Height of the tank
- A.....Area of tank
- $\frac{dx}{dt}$ .....The derivative of x concerning the time
- R.....Resistance flow
- Cm.....Cent meter
- $\omega_b$ .....Low frequency
- $\omega_h$ .....High transition frequency

LIST OF ABBREVIATION

CL.....	Closed-loop
LTI.....	Linear time-invariant
SISO.....	Single input single output
MIMO.....	Mult input multi-output
TF.....	Transfer function
PI.....	Proportional plus integral
PD.....	Proportional plus derivative
PID.....	Proportional plus integral plus derivative
FOC.....	Fractional order controller
FOPID.....	Fractional order proportional plus integral plus derivative
TID.....	Tilt-Integral Derivative
IOPID.....	Integer order proportional integral derivative
FOPDT.....	First-order plus dead time
LQR.....	Linear quadratic regulator
ISE.....	Integral square error
ZN.....	Zeigler Nichols
CV.....	Control variable
SP.....	Setpoint
PV.....	Process variable
MV.....	Manipulated variable
BIBO.....	Bounded input bounded output
FOTF.....	Fractional-order transfer function
OL.....	Open-loop
TF2SS.....	Transfer function to system space

LQ.....Linear quadratic  
ARE.....Algebraic Riccati equation  
ODE.....Ordinary differential equation  
MATLAB.....Matrix laboratory  
FOMCON.....Fractional-order modeling control

## ABSTRACT

Process control is essential in the industrial process because it guarantees the safety and optimization in a process. Additionally, process control is a useful tool to satisfy the environmental procedure and product quality necessities. In industries, one of the controlling process variables are level process, the liquid level controllers are a significant concern popular process & collective illustrative also real-world in engineering methods, liquid level coupled tank system, and can be arranged in a fashionable form of interacting and non-interacting, in this thesis work only focus on interacting coupled tank control systems, industries there many problems which affected the liquid level, so to overcome this problem and obtain constant stable output and fast response various controllers are required.

In the automated function of the interacting container, the scheme is commonly recycled exclusively in the level process. A mechanism to regulate the process height in container and movement among tanks are problematic in the process automation. Industrial engineering requires fluid moved and reserved in a container, and then stirred to another to the container. This thesis work presents the improvement concerning fractional order (FOPID) and Determinate the Optimal controller for governing a desired liquid level of the tank system. Various fractional-order (FOPID, Crone, TID, and FOLL) controller and control techniques will be tested to get a great performance. The output response is conducted within MATLAB®/Simulink® situation to verify the performances of the system in terms of rising Time ( $T_r$ ), Settling Time ( $T_s$ ), and Overshoot (OS). From the controller included in this thesis work the Linear quadratic regulator with integral action (LQI) controller have better performance compare to the other, the overshoot 7.690% and rise time 218.48 msec.

*Keywords:* -Process control, Level process, coupled tank, proportional integral derivative (PID), Fractional-order-controller Proportional-Integral-Derivative (FOPID), Linear Quadratic-Regulator (LQR).

## CHAPTER ONE

### INTRODUCTION

#### 1.1.BACKGROUND

Many industrial and scientific processing needs an understanding of the amount of the contents in the tank and other containers. In many cases, it is not conceivable or not real-world to directly view the inner. The more understandable industrial application includes tank level gauging of milk, beer, wine in the industry, level gauging of acid, oil, water treatment, and solvent vessels in chemical plants; level checking of liquid in reservoirs. Done the last several decades, computer control of the manufacturing system has been the focus of the general investigation.

As a significant consideration of fashionable method regulator, impression fluid height proceeding making cannot remain overlooked, because safety, actual and appropriate regulatory process heights are essential. Controlling of height process regulator is the main problem in a process variable along with a mutual demonstrative & concrete in engineering practices. The preceding manufacturing, human-contained the fluid height terminated a permanent fluid height adjustment while a liquefied corresponding with a convinced level; now this thesis work paper, the success of both fractional-order and linear-quadratic a controller to control the process level concerning the interacting container is analyzed. The multitude of industries processes such as petrochemical, water treatment, and beer is with a container to govern the height. A fluid height necessarily governed within appropriate controllers. A key purpose of a controller in the system keeps a height at the chosen set point and adaptable to the new set-point [1].

The integer-order PID controls are normally recycled in controlling a height the container in industries since governing the liquid in many tanks and undertaking between the tanks is a straight forward complication in the industries. Proportional controllers are appropriate to lead industrial controllers and for that reason are issues of steady exertion for the progress of their performance and robustness. One of the prospects to increase proportional integral derivative controls are to practice  $PI^\lambda D^\mu$  among fractional-order integer portions. It permits us to adjust derivative  $Miu(\mu)$  and integral  $\lambda$  order in addition to the  $K_p$ ,  $k_i$  and  $k_d$  constants where the values of  $\mu$  and  $\lambda$  invention between zero and one [1]. The process industries needed liquid to stand pumped and stored

fashionable the tanks and formerly pump it to another tank. Repeatedly the level of the tank must be controlled. In furthermost industries, chemical processes present many stimulating outstanding to their non-linear dynamics behavior.

Numerous schemes are nonlinear in type as well as such type of plant control is very tedious. In such systems, a proper controller can be designed through linearization techniques, and then the linearized system is easily controlled by two controllers such as fractional order controller and the LQR controllers [2]. In earlier industry, people controlled the liquid level done a fixed liquid height switch. When the liquid is up to a convinced level, the switch is mechanically closed or open to control the level. With the continuous progress of mechanization, continuous control of the liquid level, that is, perceiving the fluid level at all a time is required in the industry. From the time when the non-linearity, the controller develops a challenging task to attain reasonable performance using different shapes of tanks. The non-linearity rises outstanding to its shape by way of next to the end, it is broad and at the lower end, it becomes thin.

The primary task of the controls to retain a scheme at a setpoint alternatively, from a FOC, FOPID controller which is widely applied in feed-back control of the industrial system. These controllers are defined with their simple construction and method. FOPID controllers are also enabled to provide expected performance for systems [3]. This thesis work is intended to analyze the achievement of FOC like the FOPID, Crone, TID, and FOLL, controller on process level control. In this study, I chose the LQ Regulator and a FOPID controller for level process control in coupled tanks then correlate a performance response of two investigators, after that will select which controllers have a better performance for process-level control in coupled tank systems.

## 1.2. STATEMENT OF THE PROBLEM

The statement of the problem is to investigate the performance comparison of FOPID controller and LQR for process-level control in a combined tank in an instance of the interacting method & also control the external disturbance of the system, there is a complication allow analogous a height appropriate to fluid popular the reservoir & move among tanks within a selected controller those are fractional order controller those are (fractional order proportional controller (FOPID) and optimal control techniques.

A particular explanation to succeed in the achievement of the systems is to applying the fractional-order and LQR controller. Now there two-controller that have been applied to

control the system, In this project, the Fractional-order and LQ-Regulator monitor will be selected by way of the feed-back detective for the process. Finally, examine the performance of the systems.

### 1.3. MOTIVATION

The improvement of the control process for coupled tank systems is complex and more stimulating because of dynamics or non-linear by nature which shows maximum point behavior. The interacting containers are a challenge as a result of the following issues.

- Non-linear & maximum stage system.
- Generally, multi-variable description effects collaboration within a coupled tanks extremely liquid could move either two direction
- A level characterized by the coupled tanks has to be conserved at the chosen setpoint.

Performance analysis of the process has a substantial problem in control as well as increasingly accepted as an area of importance in many processes industry. This problem has individual importance among the engineering community because it offers a solution to control the processes that cannot be designated by a SISO model. This thesis aims to compare the assessment of the FOPID and LQR controllers in order to get the process-level to regulate when the system is interacting.

### 1.4. OBJECTIVE

#### 1.4.1. GENERAL OBJECTIVE

The main purpose of this study is to evaluate the performance of the FOPID controllers and LQR for process-level controls in the face based on interacting systems.

#### 1.4.2. SPECIFIC OBJECTIVES

The specific objectives in order to address the general objective of the study are:

- To develop a mathematical model of the interacting tank and control the system.
- To develop the numerical modeling of both controllers.
- To design a Fractional-order controller as well as LQR for level process control systems in case of the coupled interacting system.

- To simulate the overall system using MATLAB/SIMULINK software for evaluation of the system performance.

### 1.5.RELEVANCE OF THE STUDY

In the modern industrial process controlling system is the concentration on its performance. Due to the problematic of numerous factories that interrupt the system, so to reduce the disruption and increase the quality of the product and reduced operating cost it is beneficial to analyze the attainment of a system by using Fractional-order and linear - Quadratic controller for level process control in case of coupled interacting tank system.

In this thesis, an optimal controller called the linear- quadratic and fractional-order controllers are proposed. A technique using a linear- quadratic regulator and fractional-order proportional -integral -derivative controller and also the evaluation of both the controller performance is investigated with MATLAB<sup>®</sup>/SIMULINK<sup>®</sup> simulation.

### 1.6.SCOPE OF THE STUDY

The main opportunities of this study are to analyse and compare the performance of the scheme using fractional-order PID controllers and (LQ) regulators, all of the models are implemented through simulation using MATLAB<sup>®</sup>/SIMULINK<sup>®</sup> software. The effectiveness of both controllers is studied in detail of the transient response within a chosen value of input value.

### 1.7.METHODOLOGY

The following methodologies have been used for the completion of this thesis work:

- Various works which are related to this thesis are studied.
- A numerical modeling a plant is established and a Fractional-order PID controller and linear-Quadratic regulator designed for process-level control in the case of the coupled-interacting tank system.
- The proposed system model is developed and analyzed through simulation of MATLAB<sup>®</sup>/SIMULINK<sup>®</sup> software.
- A response of both controllers from imitation results investigated and the conclusion is done.

### 1.8.ORGANIZATION OF THESIS

This thesis is prearranged into six chapters:-

Chapter one presents the overall background of the study. The main parts of this section are problem statement, objective, significances, scope, and methodology of the thesis work.

Chapter two covers the literature review of previous study and related overall information about Level process control, linear Quadratic regulator, and Fractional-order controller.

Chapter three presents the mathematical modeling of level process control of the coupled interacting tank.

Chapter four deals with the designing of the Fractional-order PID and linear Quadratic-regulator controller in order to analyze their performance for level process control in coupled tanks.

Chapter five presents the outcome of the coupled-tank system simulation results obtained with and without the Fractional-order PID controller, linear-Quadratic regulator, and also analysis performance of the system finally evaluated.

Chapter six concludes the thesis work and recommendation for future work also presented.

## CHAPTER TWO

### LITERATURE REVIEW

#### 2.1. RELATED WORK

Some important concepts and previously accomplished works related to this thesis are reviewed as follows.

K. Sundaravadivu et al (2011) proposed the schematic of Fractional-Order Controller for height regulation of Spherical Container, which is formed as a First Order Plus Dead Time method near a setpoint. A reply to the considered Fractional-order-proportional-integral-derivative controller has correlated with the classical (IOPID) controllers arranged the investigational circumstance.  $API^\lambda D^\mu$  controllers are calculated consuming the decreasing of the Integral Square Error (ISE) method. This mechanism action a real-world & analytical technique of the controller's design for the considered class a the controller's plant [4]

Mostafa.A. Fellani et al (2015) studied the industrial application of the Coupled Tank System is widely used especially in chemical process industries. The control of liquid level in tanks and flow between tanks is a problem in the process technologies. The process technologies require liquids to be pumped, stored in tanks, and then pumped to another tank systematically and development of the Proportional-Integral-Derivative (PID) controller for controlling the desired liquid level of the CTS. Various conventional techniques of PID tuning method will be tested to obtain the PID controller parameters.

Dharam et al (2016) carried out that control height a tank is an ultimate concern in the mechanism of control. The imperative a force up liquid & next to a certain dominance them indoors containers & afterward a particular conversation to another tank. Generally, the molten would remain set through biochemical responses or blending action inside affecting a container, now somewhat situations, regularly an altitude of fluid must be governed. Accordingly, fluid regulators are a dynamic movement in systems. Conical reservoirs acquire solicitations in several processes. The shape permits an improved run of fluids. Extremely, the height governor tanks are an exact problematic topic as nonlinear & continuously illustrations an adjustment happening the area [5].

S. Nagammai et al (2016) proposed the scheme of optimal control for a three-tank process a typical to control the system is very difficult in controller design because of assumed

non-linear flow and interaction between tanks. The state feedback controller (SFB) with a controller yields better achievement related to another controller. The transient response specifications and performance indices are compared and which indicates the fact of the FSFB over LQR[6].

G.Ganesh Naidu et al (2016) studied the advantage of a crone controller for level processor level process system first and second-generation crone controlling approaches used for scheming fractional-order controllers. The crone-controller is designed using the open-loop constraint which means it is based on the posterior synthesis method. The first and second-generation crone controller applied to the level process system and compared it with the integer-order PID controller. By comparing the step, frequency response, we can say the crone-controller is giving better performance compared with the conventional proportional integral derivative controller[7].

AmrutaS.jondhale et al (2015) carried out the level control of a tank system using the PID controller, in this paper to perform and control the system by using tuning the parameter of the PID controller gain, open-loop tuning method, closed-loop tuning method, and cohen-coon method[8].

Sasidharan et al (2017) performed the control of the fluid level in the tank and movement in between the tanks is a fundamental issue in industries. The liquid process requires be pumping, reserving in the tank. The liquids will be handled many times in the industries but continuously the level of fluids in the container must be adjusted. It is essential to understand how the tank is controlled and how the level control problems are solved [8].

P. Siva Sankar et al(2017) studied the performance analysis of the FOPID controls considering a multi-tank, The sectional-order-PID regulator is an expansion to the integer-order PID controller which consists of  $\lambda$ ,  $\mu$  certain forward within  $K_p$ ,  $K_i$ ,  $K_d$ . In many cases, the sectional-order PID has verified additional efficiently the traditional PID controller. A fractional-order-PID supervisor affords outstanding startup reply & a chosen effective feedback[9].

Scholar (2018) et al studied Level control processes that are usually controlled based on the error signal. They have an upstream or downstream control valve and are drained by free fall or using a pump. After the inlet flow is used for controlling the level inside the tank, the output flow represents the main disturbance in the control system for such a case

to design a rectilinear quadratic regulator and FOC and analyze the enactment of two controllers [10].

Kheleswarada(2017) et al performed the analytical modeling of the two-tank system, and also analyze the performance of a system without a controller with trial and error method by changing the position of the resistance(valve)[11].

Generally, I review the paper related to my title, their similarity and also the difference when revising the above paper, the similarly the level a tank are modeling analytical, some of the paper designs a controller for the control the system, know when analysis the gap between the revising paper and performance comparison of fractional order Proportional Integral Derivative and LQ regulator considering the height of control in the coupled tank system, the controller gain is investigated by trial and error method(tuning) and some paper have lack of controller design and also correlate with other controller but when control the level the coupled tank system using fractional order controller like( FOPID, TID fractional lead-lag compensator) and linear quadratic regulator, I have designed a controller with a specific value, performance assessment, and compare the performance of controller used in terms of transient response, eventually selected which controller has a better performance.

## 2.2.PROCESS CONTROL

Process control is essential in the industrial process because it guarantees the safety and optimization in a process. Additionally, process control is a useful tool to satisfy the environmental procedure and product quality necessities. For chemical engineers, process control is widely appropriate in the industrial process. In many processes, such as petrochemical, paper, and water treatment engineering is using the container to govern the height of the fluid. Anequitable of the controller in the level control is to maintain at a given level of the set-point and to receive a recent set-point. The proceeding industry necessitates liquid prospect inflated, deposited containers, then inflate to an alternative reservoir, the fluid handled via fraternization behavior fashionable a container, nevertheless continually the height of fluid are reservoir essential organized, besides the movement a fluid commitment remain regulated. Height & outflow governor chamber is by the side of a core industries' environments. Interacting tank (coupled tank) regulation of system manufacturing is an inspiring assignment for frequent sources outstanding to non-linearity. Controls the fluid height in the coupled container are the most important concern

in the system. The height of the fluid is too high for various disappointed response stability, cause destruction. When the liquid height is low, it might require unstable significances for the system. The industrial process presents many challenging control problems due to their non-linear dynamics behavior. Nonlinear models are used where accuracy over a wider range of operations is required where they can be directly incorporated into the controller algorithm. Because of the characteristic nonlinearity, most of the chemical process industries require control techniques, those to control such systems used are fractional order controller and linear quadratic regulator. The nonlinear system taken up for the study is the coupled tanks [11], [12]. The engineering application of liquid level control is tremendous, specifically in chemical process industries. Generally, the liquid level control happens in some of the control loops of a process control system. Much other industrial application is concerned with level control, may it be a single loop level control means single-input-single-output (SISO) or sometimes multi-input multi-output (MIMO). In some cases, level process controls that are available in the industries are for interacting and non-interacting tanks. But now for the two systems, I studied interacting coupled tank system to control the liquid level in tanks and flow between tanks is a problem in the process industries. Most of the industry uses different shapes of tank systems those are the cylindrical tank, conical tank, and spherical tank to control the flow of rate and liquid level. In our design process, I have taken a cylindrical tank because it is inexpensive and no product loss at the end-point and thus most efficient. Here all the tanks under a study have their type of manipulated variables and thus we can regulator a movement speed of the fluid in interacting coupled system through regulating a variable for our desired output. Often the tanks are so coupled together that the levels also interact this must also be controlled. So it is necessary to analyze the response in the interacting coupled mode of various tanks with applying step point. Two design controls system to manage the height of fluid on container [13]. Analytical forms are an explanation of a system in terms of equation [14]. A control system consists of sub-systems and processes accumulated to obtain the desired output with the desired input, given a specified input. This system illustration is actually necessary for numerous explanations such as to:-

- Controller designing for a specific system,
- Examine the effects of the process
- Estimate the development of the system
- Categorize issues that prompting a system

- Develop a mathematical model of a process
- Classify the correlation among interacting tanks.

The presented demonstrating methods could be categorized into three main. The process could require numerous inputs otherwise excitation of the system a purpose a time. The output of the variable must be calculated. There is a correlation between input & output [15]. The process control is an important part of almost every process operation. The structure of a process is frequently measured by process variables such as we manipulated a variable, measured variable, and controlled variable. The control of process variables is attained by the controller. Process control is repeatedly held responsible for dissimilar processes taking place in industries. Our objectives are to analyze the performance of the Fractional-order controller and linear quadratic regulator for process control in case of level control. Applying an in the effect of a controller construction to control a process provides us numerous benefits: - better control system yield, better performance, and improved the response of the system. In-process control loop a controller job is to influence the control system via the control controlled variable so that the value of the controlled variable (system) equals the value of the reference model. The controller is the “Brain “of the process control and also generates an alarm towards the ending (controlled variable) dependent upon a deviation between the desired point and the measured value of the controlled variable.

### 2.3. LEVEL PROCESS CONTROLS

Control the height of a process in a coupled container and flow among tanks is a basic problem in the process industries. The process industries want the liquid to be forced, stored in tanks, and then forced to another tank. Numerous times the determination of the liquid be treated by chemical or mixing treatment in the tanks, nevertheless continuously the level of liquid in the tanks requirement must be contained and outflow among containers needs controlling. The flow & level regulate a container is the brain of process industries. The liquid level tank can be arranged in the form of the interacting and non-interacting system can be controlled in different ways.

## CHAPTER THREE

### SYSTEM MODELING

#### 3.1. ANALYTICAL MODELING OF SINGLE AND COUPLED TANK SYSTEM

Mathematical modeling of level processes is characterized by large time constants and/or large time constants and time delays (dead time). In the instance of constant flow, in the stationary regime, the input and output quantities are equal.

Mathematical models are needed for these processes, to: determine the mechanism of the process, simulate the behavior, establish the control structures, and establish the control strategies [16].

##### 3.1.1. MATHEMATICAL MODELING OF SINGLE TANK SYSTEM

Modern control assumption has been established on effective representations that are characterized as differential equations (DE) in other words transfer functions (TF). This single tank scheme depending on the relationship amongst the movement of fluid within the container,  $q_{in}(t)$  & height of the liquid. Against the model, to design a feedback controls the output of the system at the desired point.

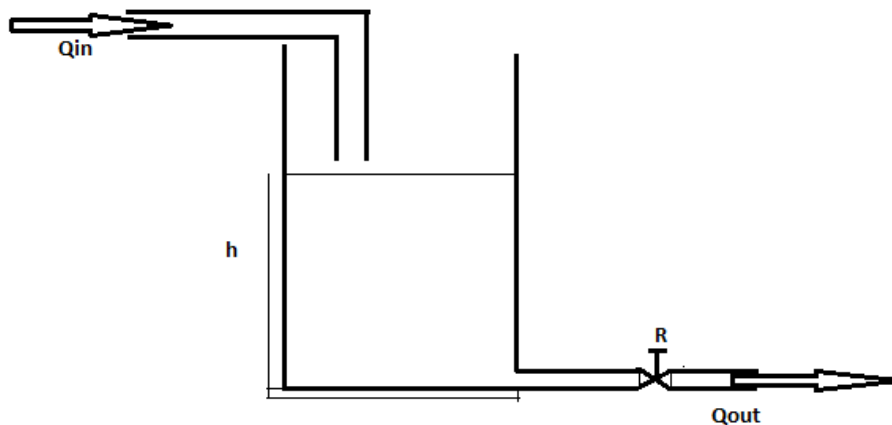


Figure 3.1:- Schematic diagram of the single tank

The liquid outflow of the tanks is equivalent to the height of fluid  $h$  if the flow rate is laminar

$$q_{out} = kh(t) \tag{3.1}$$

Where  $k$  is a constant parameter to control the flow of liquid

A mass balance across the tank gives:

$$A \frac{dh}{dt} = q_{in}(t) - q_{out}(t) \quad (3.2)$$

Where  $q_{in}$  –the flow rate of the liquid in

$q_{out}$  –the flow rate of liquid out

$A$  – Cross-sectional area of the tank

When we re-arrange the above equation

$$\frac{dh}{dt} + \frac{1}{A} q_{out}(t) = \frac{1}{A} q_{in}(t) \quad (3.3)$$

Equation (3.3) is re-considered and a new variable,  $x_1$  is currently defined

$$x_1(t) = h(t)$$

Eqn (3.3) inscribed as:

$$\begin{aligned} \frac{dx_1}{dt} \\ = -\frac{k}{A} x_1 + \frac{1}{A} q_{in}(t) \end{aligned} \quad (3.4)$$

Generally when expressed the above equation in the state-space form:

$$\frac{dx}{dt} = \dot{X} = Ax(t) + Bu(t) \quad (3.5)$$

The previous equations are in general, linear equation when  $x$  is state variables of a system. A liquid in a tank, there is a variable  $h(t)$  and a particular input  $q_{in}(t)$  variable. so the matrices  $A$  and  $B$  are scalars. So the nonlinear system can be written in the form of:

$$\frac{dx}{dt} = y(x, u, t) \quad (3.6)$$

Where  $y$  is a non-linear, function.

### 3.1.2. ARTIMETICAL MODELING OF TWO INTERACTING TANK SYSTEMS

Plant modeling [17][18]

In this case, two tanks connected to form a coupled tank system; here the level of the first tank depends on the level of the second tank. The straightforward principle of control interacting two containers is continued the height of the tank steady after the

flow of fluid into the tank and flow out of the liquid. The parameter used to control the coupled-tank system in process industries height, and cross-sectional area. To retain&governor the level liquid at an indicated value, the flow input rate is regulated [19]. The interacting (coupled) tank in figure 3.2.

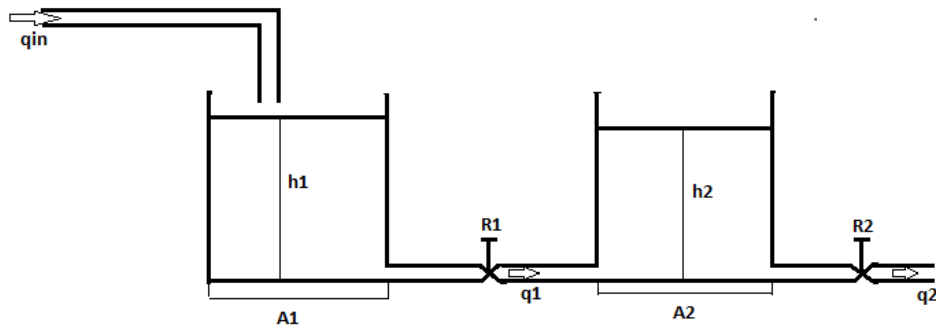


Figure 3.2:- Block diagram of interacting (coupled) tank [20]

Plant transfer function Resistance modeling, [18]

Consider the flow of the liquid through a pipe from the pump to the process tank and another pipe from the process tank to another process tank.

The resistance  $R$  for liquid-flow in such a container is defined as the change in the level difference to unit change in flow rate, that is

$$R = \frac{\text{change in level difference, cm}}{\text{change in flow rate, cm}^3 / \text{sec}} \quad (3.7)$$

Where,

$q_{in}$  = Volumetric flow rate input (  $\text{cm}^3 / \text{sec}$  )

$q_1$  = Volumetric flow rate from tank 1 to tank 2 (  $\text{cm}^3 / \text{sec}$  )

$q_2$  = Volumetric flow rate from tank output (  $\text{cm}^3 / \text{sec}$  )

$h_1$  = Height of the liquid level input (cm)

$h_2$  = Height of the liquid level in tank 2 (cm)

$A_1$  = Cross-sectional area of tank 1 (  $\text{cm}^2$  )

$A_2$  = Cross-sectional area of tank 2 ( $\text{cm}^2$ )

$R_1$  = Linear resistance of flow from tank 1 through valve 1 ( $\text{cm}^{-2}\text{sec}$ )

$R_2$  = Linear resistance of flow from tank 2 through valve 2 ( $\text{cm}^{-2}\text{sec}$ )

For tank 1:-

Mass balance equation can be written as

$$A_1 \frac{dh_1}{dt} = q_{in} - q_1 \quad (3.8)$$

From the definition of resistance, the relationship between  $q_1$ ,  $h_1$  and  $h_2$  is given by

$$q_1 = \frac{h_1 - h_2}{R_1} \quad (3.9)$$

$$A_2 \frac{dh_2}{dt} = q_1 - q_2, \quad q_2 = \frac{h_2}{R_2} \quad (3.10)$$

$$A_1 \frac{dh_1}{dt} = q_{in} - \frac{h_1 - h_2}{R_1} \quad (3.11)$$

$$R_1 A_1 \frac{dh_1}{dt} = R_1 q_{in} - h_1 + h_2 \quad (3.12)$$

By taking Laplace transform

Note that  $R_1 A_1$  is the time constant of the system, Taking the LT of both sides of equation (3.12), we obtain

$$R_1 A_1 s h_1(s) = R_1 q_{in}(s) - h_1(s) + h_2(s) \quad (3.13)$$

$$h_1(s) = \frac{R_1 q_{in}(s) + h_2(s)}{1 + R_1 A_1 s}, \quad \text{when } \tau_1 = R_1 A_1$$

$$\begin{aligned} & h_1(s) \\ &= \frac{R_1 q_{in}(s) + h_2(s)}{1 + \tau_1 s} \end{aligned} \quad (3.14)$$

For tank 2, the mass-balance equation is

$$\left. \begin{aligned} A_2 \frac{dh_2}{dt} &= q_1 - q_2 \\ A_2 \frac{dh_2}{dt} &= \frac{h_1 - h_2}{R_1} - \frac{h_2}{R_2} \end{aligned} \right\} \quad (3.15)$$

$$\begin{aligned} R_2 R_1 A_2 \frac{dh_2}{dt} &= R_1 h_1 - R_2 h_2 \\ &\quad - h_2 R_1 \end{aligned} \quad (3.16)$$

On dividing by  $R_1$  and taking Laplace transform

$$A_2 R_2 h_2(s) + \frac{R_2}{R_1} h_2(s) + h_2(s) = \frac{R_2}{R_1} h_1(s) \quad (3.17)$$

$$h_2(s) \left( \tau_2 s + \frac{R_2}{R_1} + 1 \right) = \frac{R_2}{R_1} h_1(s) \quad (3.18)$$

Where  $\tau_2 = R_2 A_2$

$$\begin{aligned} h_2(s) \left( \tau_2 s + \frac{R_2}{R_1} + 1 \right) \\ = \frac{R_2 R_1 q_{in}(s) + h_2(s)}{R_1 (1 + \tau_1 s)} \end{aligned} \quad (3.19)$$

$$\begin{aligned} h_2(s) \left( \tau_2 s + \frac{R_2}{R_1} + 1 \right) (R_1 + R_1 \tau_1 s) \\ = R_2 R_1 q_{in} + R_2 h_2(s) \end{aligned} \quad (3.20)$$

$$\begin{aligned} h_2(s) \left( \tau_2 s + \frac{R_2}{R_1} + 1 \right) (R_1 + R_1 \tau_1 s) - R_2 h_2(s) \\ = R_2 R_1 q_{in} \end{aligned} \quad (3.21)$$

Therefore the transfer function of the two tanks interacting system is expressed as

[21] , [22].

$$\begin{aligned} \frac{h_2(s)}{q_{in}(s)} \\ = \frac{R_1 R_2}{R_1 \tau_2 \tau_1 S^2 + (R_1 \tau_1 + R_1 \tau_2 + R_2 \tau_1) S + R_1} \end{aligned} \quad (3.22)$$

$$= \frac{R_2}{\tau_2 \tau_1 S^2 + (\tau_1 + \tau_2 + R_2 A_1) S + 1} \quad (3.23)$$

Accordingly, effecting state-space Eqn. of a plant can be imitated from the above Eqn(3.23)

$$\begin{cases} \dot{x} = Ax + Bu \\ y = Cx + Du \end{cases} \quad (3.24)$$

$$\dot{X} = \begin{bmatrix} 0 & 1 \\ \frac{-1}{\tau_1 \tau_2} & -\left(\frac{\tau_1 + \tau_2 + R_2 A_1}{\tau_1 \tau_2}\right) \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{R_2}{\tau_1 \tau_2} \end{bmatrix} [q_{in}] \quad (3.25)$$

$$y = [1 \quad 0] \begin{bmatrix} X_1 \\ X_2 \end{bmatrix} \quad (3.26)$$

$$A = \begin{bmatrix} 0 & 1 \\ \frac{-1}{\tau_1 \tau_2} & -\left(\frac{\tau_1 + \tau_2 + R_2 A_1}{\tau_1 \tau_2}\right) \end{bmatrix} \quad (3.27)$$

$$B = \begin{bmatrix} 0 \\ \frac{R_2}{\tau_1 \tau_2} \end{bmatrix}, C = [1 \quad 0], D = 0 \quad (3.28)$$

When the interacting coupled tanks with disturbance system

Disturbance Analysis for an interacting container of fluid height control systems can be as follows:-

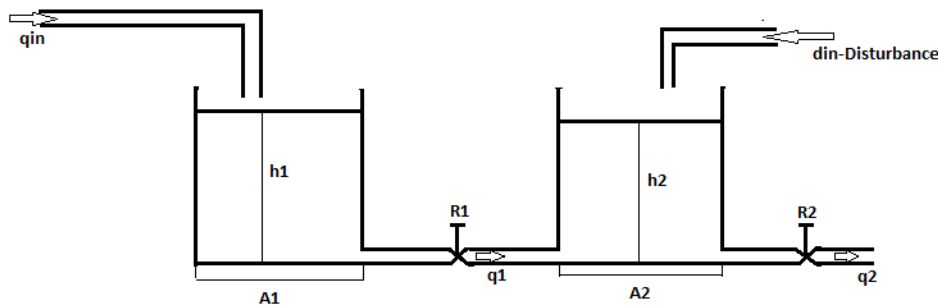


Figure 3.3:-Block diagram of the interacting two with a disturbance

The disturbance transfer function analysis as below

$$A_2 \frac{dh_2}{dt} = d_{in} - q_2 \quad (3.29)$$

Since from equation (3.7),  $q_2 = \frac{h_2}{R_2}$

$$\begin{aligned} A_2 \frac{dh_2}{dt} \\ = d_{in} - \frac{h_2}{R_2} \end{aligned} \quad (3.30)$$

The LT of equation (3.30)

$$\frac{H_2}{D_{in}} = \frac{R_2}{R_2 A_2 s + 1} \quad (3.31)$$

## CHAPTER FOUR

### CONTROL METHODS AND DESIGN

#### 4.1. INTRODUCTION

A control structure is a system or set of devices, that manages the signal or regulates the presence of additional processes or systems to accomplish the chosen set-point. In other words, the definition of the control method can be simplified as a system, which controls other systems, two forms of control mechanism are: -Open and closed-loop control method. Open-loop control methods are wherein a control accomplishment is self-regulating the response of the processes but a (CLS) in which the response affects the input to insure that the input will regulate itself based on the response generated [23]. The closed-loop is summarized below in the tabular form.

Controller	Advantage
Closed Loop	Disturbance rejection
	Guaranteed attainment uniform within prototypical uncertainties, while the structure does not much faultlessly the physical process and model specification are not exact
	The unbalanced process can be steadied
	Compact feeling to strict differences
	Improved performance using reference tracking

	Increased productivity
	Energy and Material Conservation
	Safety

Table 4.1:- summarization of closed-loop controller

If the process does not happen the desired performance specification, Controllers are used & also connected either in series with or parallel to the plant depending upon the specification. A show figure 4.1 feedback control system with a controller.

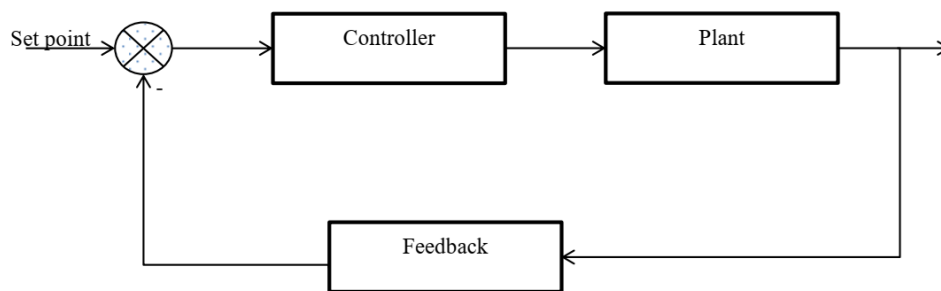


Figure 4.1:- closedControl system

As shown above, figure controls signals generated which are the change among the set-point signal and the feedback signal. The control signal resolves the scale through whatever an output response returns against the desired point. The controller is staying to keeping the controlled variable such as liquid level, motor speed; robot joint angle at a set-point in addition to the input signal is applied to plant which will give good results. Feedback control is an action in which a measured variable is compared to its desired value to produce an error signal which is acted upon in such as to decrease the amplitude of the error. For a plant with MIMO, the situation desires numerous regulators. For a SISO, need a single controller for managing.

#### 4.2. TYPE OF CONTROLLERS

There is various type of controller used for performance analysis of the system within a given specification. Controllers can be categorized into, feed-back, & feed forward regulator. A contribution near the feed-back controls identical as anything frustrating toward governor output regulates are feed-back interested in the supervisor. The feed-back method frequently affects in-between some place well-ordered

variables aren't preferred setpoint. The feedforward governor put up escape slow-ness a feed-back regulate through feedforward controller.

proportional – integral – derivative controller  
 pole – placement controller  
 optimal controller
 } are feedback controller

A first two-controller is the feedback controller & A third is an FSB control. Pole-placement regulators are a feed-back controller that used for placing closed-loop-poles to the setpoint position in the plane. Pole-placement used only for (SISO) system. For the (MIMO) multiple-input multiple-output system, the complication of over-abundance of the designed controller is challenged, for such one system, we didn't recognize in what way to define all the construct elements, since simply a restricted amount of system can be initiated on or after the closed loop-pole position. A pole-placement-technique some trial & error technique with pole-position was a prerequisite for we don't recognize previously whatever pole location will contribute better achievement. The controller is equipment introduced to monitor the process and adjust some variables to maintain the system at or near desired conditions. Controller design has a significant place in the region of industrial engineering. The type of controller to be selected is important for achieving the desired design criteria. In practice, simple and low-degree controller structures are preferred most of the time. Among these controller structures, fractional-order controller those are fractional-order proportional-integral controller, crone controller, TID controller, fractional-order lead-lag controller, and LQ regulator is desired the most. There are two varieties of controllers are selected to evaluate the performance of process-level control in form of the interacting in this thesis work. Those are fractional order controller (FOC) and linear quadratic controller (LQR). Then the application of those two different controllers in industrial applications is compared [14],[23]

The overall importance feature of feedback control analyzed as follows:

Controller	Importance feature of feedback control
Feedback control	To stabilize the system
	To reduce system output sensitivity to system parameter variation due to the established of the system components.
	To reduce the impact of external a system effect on the system response

	To improve the system output transient
	To improve the system output steady-state response
	To reduce system sensitivity inaccuracies in the system measurements

Table 4.2:- importance features of the feedback controller

#### 4.2.1. INTRODUCTION PID CONTROLLER

PID (proportional–integral–derivative) controller used as a linear controller for the control process with a good response. A PID controller is commonly used in the process and applying accurately the grouping of three types of corrective actions to the error signal, which characterizes how far or near is the desired set point from the actual output. As widely known, these three control actions are proportional, integral and derivative. The key feature when tuning PID controllers is in determining how to best syndicate those three terms to accomplish the most well-organized instruction of the process variable for the well-thought-out problem. As well known, the most understandable way is to use a simple weighted sum where each term is multiplied by a tuning constant or gain [24]. PID's working principle is that it calculates an error value from the processed measured value and the desired reference point. The work of the PID-controller is to reduce the error by changing the inputs of the system. Proportional-integral derivative control is the basic control scheme of the classical control system. [25]. The assessment of a process could be improved [26] using the proper value of gain  $k_p$ ,  $k_d$ , and  $k_i$ . An analytical equation for (IOPID) control of a system within a value of input  $u(t)$ , value of output  $y(t)$  and error signal  $e(t)$  is communicated as (4.19) & (4.20) where  $k_p$  – is the proportional gain,  $k_i$  – integral gain, and  $k_d$  – derivative gain.

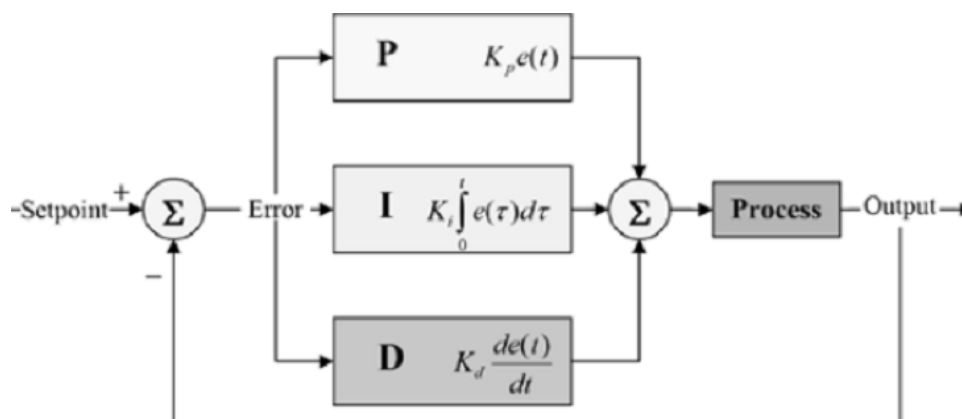


Figure 4.2:- PID Controller [27]

Proportional gain  $k_p$  –improve the structure to create it protected a capacity disorder.  
 $k_i$  –Supportsto decrease errors of steady-state. $k_d$  –Supports to improve the establishment of a closed loop [26].IOPID controller is the type of feed-back whose output a control variable (CV), are built on the error among a setpoint (sp) and process variable.

$k_p$  –is responsible for system stable moreover low and the process variable can implication; moreover high and a process variable can oscillate.

$k_i$  –Accountable for a fault to zero, but toward established  $k_i$  too great is to request vibration.

$k_d$  –Is accountable for a plant output to great & the process variable will vibrate. Proportional-integral-derivative-controller challenges to accurate the inaccuracy among an (MV) & the setpoint value by computing. Error is the reference to regulate some input to the system to a well-defined set-point. The output of the controller can be defined in terms of the receptiveness of the controller to an error signal, the amount to which the controller overshoots the set-points and the amount of system oscillation [26].

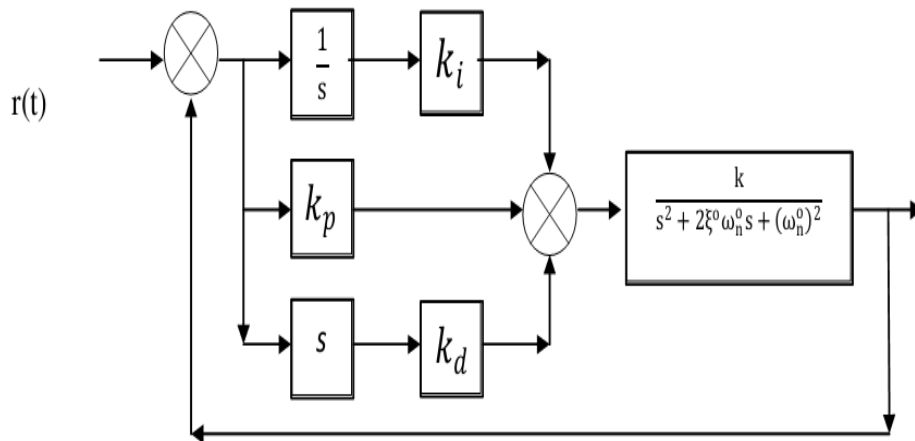


Figure 4.3:- PID controller design for a system

The input of PID controller  $r(t)$  is equivalent to the sum of the three signals, the signals obtained by multiplying the error signal by a constant proportional gain plus the signal obtained by differentiating and multiplying the error signal by constant integral gain [28]. The output of the PID controller is given Laplace to transform and it is given as

$$r(t) = K_p e(t) + K_d \frac{de(t)}{dt} + K_i \int e(t) dt \quad (4.1)$$

The three terms of integer order (IOPID) controls have three modes of a control character. Those three basic modes are -proportional, integral, and derivative terms. IOPID-the regulators are used in monitoring the closed-loop practice open-loop process. Controller parameterized mathematically as:-

Taking Laplace transform

$$G_C(s) = \left[ k_p + \frac{K_i}{s} + k_d s \right] \quad (4.2)$$

Where

$K_p$  – Proportional gain

$K_i$  – Integral gain

$K_d$  – Derivative gain

$$G(s) = K_p(e(t) + \frac{1}{T_i} e(t) + T_d \frac{de(t)}{dt}) \quad (4.3)$$

Where  $T_i$  – is the integral is time and  $T_D$  – is the derivative time, The P-parts entertainment arranged current value of the error signal, the integral -represents a normal of aforementioned miscalculations and an imitative can assumed a forecast of forthcoming fault constructed on linear-extrapolation well-meaning of note that the control-unit(t) are fashioned exclusively starting the inaccuracy  $e(t)$ . Generally, in a process, the PID controllers are terms donates and accomplishing transient response & stability [28], [29].

		Effect of Performance			
Term	Response	Settling time	Rise time	Overshoot	Steady-state error
Proportional	$K_p$	Small change	decrease	Increase	Decrease
Integral	$K_i = K_p \left(\frac{1}{T_i}\right)$	Increase	decrease	Increase	Eliminate

Derivative	$k_d = K_p T_d$	Decrease	Small change	Decrease	Small change
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Table 4.3:- Effect of the parameter in a closed-loop system

Where  $k_p$  denotes the proportional gain,  $k_i$  denotes the integral gain, and  $k_d$  denotes the derivative gain correspondingly. The proportional term determines the change of the yield that is comparative to the present inaccuracy. The comparative gain duration is disturbed within the obtainable state of the variable. The important gain term is comparative for mutually the greatness and interval of an error. The comparative term increases the speed of the process towards the set-point & reduces the remaining steady-state error that could happen within a proportionate only regulator. The aggregate of a difference of the progression error is intended by responsible the variance in line of the derivative concerning interval. Ratio of a transformation of the error is reproduced via the  $k_d$  – gain. Classical PID regulators have been extensively used in industry, due to their effectiveness for linear systems, ease to understand, and simple to implement. As shown in a figure, the actual controller of the process can be investigated by the transient response.

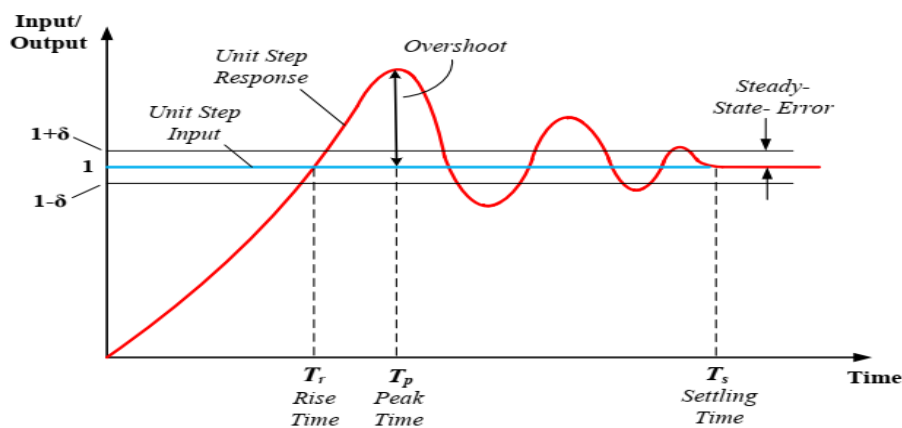


Figure 4.4:- parameters of transient response analysis of a controller

#### 4.2.1.1. COMPARATIVE-TERM

A comparative term is formed through a yield assessment to the stimulating error significance. The comparative response is attuned by enlarging the error by a relentless improvement called the comparative gain constant.

The proportional term is given by:-

$$\left. \begin{aligned} u(t) &= k_p e(t) \\ \text{or} \\ \frac{(S)}{E(S)} &= K_p \end{aligned} \right\} \quad (4.4)$$

An extraordinary  $K_p$  result is happening a large difference output for the particular modification in the inaccuracy. When the gain of proportionality is high, the process is unsteady, whereas an insignificant yield reply to an enormous input error & a less reaction to an oversized contribution is small, the control accomplishment might stand beside unimportant as soon as replying toward the process instabilities. In the Manufacturing preparation, the comparative term indicates that too packaged of the output adjustment.

#### 4.2.1.2. INTEGRAL TERM

An effect of integral gains are comparable to together the amount of the error & the interval of error. A PID controller, the essential gain are sums of immediate inaccuracy over interval & contributes the collected off-set that would need been modified formerly. Gathered inaccuracy is then increased by the fundamental improvement ( $K_i$ ) and further to the regulator output.

Basic is expressed as follow:

$$u(t) = k_p e(t) + k_i \int_0^t e(\tau) dt \quad (4.5)$$

Or

$$\begin{aligned} \frac{u(s)}{e(s)} \\ = k_p \left( 1 + \frac{1}{\tau_i s} \right) \end{aligned} \quad (4.6)$$

#### 4.2.1.3. DERIVATIVE TERM

Imitative of a system inaccuracy is premeditated by responsible the gradient of the inaccuracy over interval & aggregate percentage of deviation by the  $k_d$  relentless.

A Derivative term is set by

$$u(t) = k_p e(t) + k_d \frac{de(t)}{dt} \quad (4.7)$$

Or

$$\frac{u(s)}{e(s)} = k_p (1 + \tau_d s) \quad (4.8)$$

The function of the derivative term slows the rate of variation of the controller response to control the system plant. The derived controllers are used to decrease the level of the overshoot designed by the integral element and increase the combined controller system stable. Conversely, the derivative term reduces the momentary response of the regulator, and similarly, differentiation of a signal increases sound and thus this term in the controller is highly sensitive to noise in the error term and can source a process to become unstable if noise and the derivative improvement are sufficiently large.

In general, The PID controller has the following form in the time-domain, The Overall Characteristics of Proportional, Integral, Derivative Controller gain in Tabular Form.

Proportional control	$u(t) = K_p e(t)$
Integral control	$u(t) = K_i \int e(t) dt$
Derivative control	$u(t) = K_d \frac{de(t)}{dt}$

Table 4.4:- Features of the PID controller

Steps for the scheme a Proportional-integral-derivative controller are:-

Step1: determine each value of PID controller gain within the desired system

Step2:- decide what features of the process essential developed.

Step 3:- used comparative gain to reduce a rise-time

Step4:- use derivative gain to decline the overshoot and settling time

Step5:- use integral gain to remove the steady-state-error.

Proportional-integral-derivative (PID) controllers are the feedback loop controlling mechanism. It corrects the error between a measured variable value and the desired set-point by calculating and then a corrective action adjust the process (plant) as per the

requirement. The biased addition of these three arrangements is used to adjust the process (plant) via a control element. By calculating the value of three constants in the PID controller, it can offer a control achievement designed for specific activities.

The overall structure of the system control is shown in Figure 4.5.

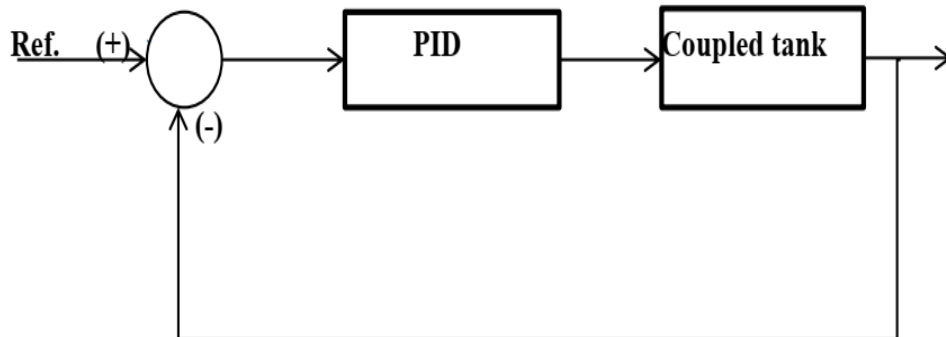


Figure 4.5:- closed-loop control systems

Now to design the proportional-integral-derivative controller a certain dominant pole placement method is used. The transfer-function of PID controller need to Control 2<sup>nd</sup> - order plant ( $G_p(s)$ ) (with slow “S” designed or oscillatory open-loop changing aspects).

$$G_C(s) = \left[ k_p + \frac{K_i}{s} + k_d s \right] = \frac{k_d s^2 + k_p s + k_i}{s} \quad (4.9)$$

Where,

$$\left. \begin{array}{l} K_p - \text{proportional gain} \\ K_i - \text{integral gain} \\ K_d - \text{derivative gain} \end{array} \right\} \text{ of the PID controller}$$

In general, a second-order plant is categorized by an open loop transfer-function,

$$G_p(s) = \frac{K}{s^2 + 2\xi\omega_n s + \omega_n^2} \quad (4.10)$$

Where,

the  $\xi$ -damping-ratio of a second-order plant

$\omega_n$ -The natural- frequency of a second-order plant

K-Constant of a 2<sup>nd</sup> -order plant

Then, the CL transfer function becomes,

$$G_{c1}(s) = \frac{G_c(s)G_p(s)}{1 + G_c(s)G_p(s)} \quad (4.11)$$

Substituting for  $G_c(s)$  and  $G_p(s)$  from Equation (4.8) and (4.10) respectively into Equation (4.11) gives,

$$G_{c1}(s) = \frac{k(k_d s^2 + k_p s + k_i)}{S^3 + (2\xi\omega_n + k k_d)S^2 + (\omega_n^2 + k k_p)s + k k_i} \quad (4.12)$$

From the equation (4.10) perfectly, the open-loop plant has two poles at  $[\xi\omega_n \pm j\omega_n\sqrt{1 - \xi^2}]$  and from (4.11) are also obvious the PID-controllers have one extreme at derivation & also two-zeros at  $\left[\frac{K_p}{2K_d} \pm j\sqrt{\frac{K_i}{K_d} - \frac{K_p^2}{4K_d^2}}\right]$ . Dignified that together the poles of the system & regulator zero are complex conjugates in the complex s-plane. From Eqn (4.12), assumed that the CL classification has two zeros & three poles in the complex s-plane and position of a CL zeros continuemodest as in (4.11) although the location of CL poles differences dependent on the PID controller gains. For definite full state feedback with PID- controllers, the CL system (4.12) would require one real-pole which necessity be extreme away since the real part of the additional two complexes (conjugate) CL poles as considered [30]. Currently, if the chosen CL calculations of a 2<sup>nd</sup>-order scheme can be specified as the qualifications on  $\xi^{c1}$  (Damping ratio of the CL scheme) and  $\omega_n^{c1}$  (Natural frequency of the CL system) as in one can simply substitute the location of the real zero ( $\alpha$ ) by  $(-\alpha\xi^{c1}\omega_n^{c1})$ , providing that  $s = -\alpha\xi^{c1}\omega_n^{c1}$  is chosen to be huge sufficient regarding  $(-\alpha\xi^{c1}\omega_n^{c1})$ . Consequently, selecting an appropriate of comparative governance ( $\alpha$ ), the third-order CL system (3<sup>rd</sup>) [31] determination accomplish like second-order classification consuming the user-specified CL damping-ratio  $\xi^{c1}$  (percentage of maximum overshoot) & closed-loop natural frequency  $\omega_n^{c1}$  (Rise- time). In this situation, the distinguishing polynomial is inscribed as:

$$(S + \alpha\xi^{c1}\omega_n^{c1})(S^2 + 2\xi^{c1}\omega_n^{c1}S + (\omega_n^{c1})^2) = 0 \quad (4.13)$$

After multiplication Equation (4.13) is written as,

$$s^3 + (2 + \alpha)\xi^{C1}\omega_n^{c1}s^2 + (1 + 2(\xi^{C1})^2(\omega_n^{c1})^2)s + \alpha\xi^{C1}(\omega_n^{c1})^3 = 0 \quad (4.14)$$

Relating the coefficients of Equation (4.13) with the denominator of Equation (4.12), mathematical modeling of PID controller gain can be calculated as follow.

$$\omega_n^2 + kk_p = 1 + 2\alpha(\xi^{C1})^2(\omega_n^{c1})^2 \quad (4.15)$$

$$k_p = \frac{(1 + 2\alpha(\xi^{C1})^2(\omega_n^{c1})^2) - \omega_n^2}{K} \quad (4.16)$$

$$kk_i = m\xi^{C1}(\omega_n^{c1})^3 \quad (4.17)$$

$$k_i = \frac{\alpha\xi^{C1}(\omega_n^{c1})^3}{K} \quad (4.18)$$

$$2\xi\omega_n + kk_d = (2 + \alpha)\xi^{C1}\omega_n^{c1} \quad (4.19)$$

$$kk_d = (2 + \alpha)\xi^{C1}\omega_n^{c1} - 2\xi\omega_n \quad (4.20)$$

$$k_d = \frac{(2 + \alpha)\xi^{C1}\omega_n^{c1} - 2\xi\omega_n}{K} \quad (4.21)$$

The overall PID gains are:-

$$\left. \begin{aligned} K_p &= \frac{(1 + 2\alpha(\xi^{C1})^2)(\omega_n^{c1})^2 - \omega_n^2}{K} \\ K_i &= \frac{\alpha\xi^{C1}(\omega_n^{c1})^3}{K} \\ K_d &= \frac{(2 + \alpha)\xi^{C1}\omega_n^{c1} - 2\xi\omega_n}{K} \end{aligned} \right\} \quad (4.22)$$

The Four major characteristics that affect systems dynamically can be summarized in the table below

No	Transient response	Characteristics of the response
1	Rise-time	period it profits aimed at process response to rising away after 90% of a set-point significance for the first time
2	Over-shoot	In what way considerably the ultimate assessment is high than the steady-state regularized beside the steady-state
3	Settling-time	The historical its receipts for the system to touch to its steady-state
4	Steady-state-error	The modification among a steady-state & the preferred reply

Table 4.5:- Transient response for Performance analysis

#### 4.3. BASIC CONCEPT OF FRACTIONAL ORDER CALCULUS

Fractional Calculus is an old accurate topic since the 17<sup>th</sup> century. The sectional-order calculus is a zone of mathematics that expresses through derivatives and integrals since non-integer guidelines. In additional verses, it is an oversimplification of the old-fashioned calculus that indicates equivalent conceptions & apparatuses but through considerably larger appropriates. Happening the last two periods, fractional calculus has remained recollected by scientists and engineers and applied in an accumulative number of grounds, specifically in the area of regulator concept. Fractional calculus benefits to estimate  $\frac{d^\alpha}{dt^\alpha}$ ,  $\alpha$  –fold integrals where  $\alpha$  is fractional, irrational, or complex. Fractional-order systems,  $\alpha$  is measured to be fractional. These mathematical occurrences permit telling an actual purpose additional correctly than conventional integer order approaches. The foremost purpose of expending the integers order simulations remained the interval of an explanation aimed at sectional differential equivalences. The success of fractional-order controllers is absolute with a lot of success due to the development of effective methods in differentiation and integration of non-integer order equations. There are dissimilar explanations of Fractional-Order differentiations and integrations. Some of the definitions range directly from integer-order calculus. The well-established definitions include the Grunwald-Letnikov definition, the Cauchy integral formula, the Caputo definition, and the Riemann-Liouville. Classification determination stands brief as follows & formerly their belongings determination remains particular.

### 4.3.1. FRACTIONAL ORDER CALCULUS

**Cauchy's** description of fractional order integration characterization is an all-purpose the per-request of the integer-order Cauchy formulation

$$D^\gamma f(t) = \frac{\Gamma(\gamma + 1)}{2\pi j} \int_C \frac{g(\tau)}{(\tau - t)^{\gamma+1}} d\tau \quad (4.23)$$

Wherever C is the smooth curve neighboring the single-valued purpose g (t).

**Grunwald–Letnikov's** explanation is perhaps the best known due to its most suitability for the realization of discrete control algorithms. The Grunwald-Letnikov definition is expressed as [32]

$$a^{D_t^\alpha} f(t) = \lim_{h \rightarrow 0} \frac{1}{h^\alpha} \sum_{j=0}^{\lfloor \frac{t-a}{h} \rfloor} (-1)^j \binom{\alpha}{j} f(t - jh) \quad (4.24)$$

Where  $w_j^\alpha = (-1)^j \binom{\alpha}{j}$  denotes the factors of the polynomial  $(1 - z)^\alpha$  the factors can likewise remain developed recursively beginning

$$w_0^\alpha = 1, w_j^\alpha = \left(1 - \frac{\alpha + 1}{j}\right) w_{j-1}^\alpha, \text{ when } j = 1, 2, 3, \dots$$

**Riemann–Liouville** defines Fractional Order differentiation the Fractional Order integration is defined as [33]

$$a^{D_t^{-\beta}} f(t) = \frac{1}{\Gamma(\beta)} \int_b^t (t - \tau)^{\beta-1} f(\tau) d\tau \quad (4.25)$$

Where  $0 < \beta < 1$  and  $b$  - the leading interval design, repeatedly supposed towards remain nothing, i.e.,  $b = 0$ .

The separation is formally designated as  $a^{D_t^{-\beta}} f(t)$

The explanation exceeding formula is the nethermost repeatedly recycled classification in fractional- order calculus. The indexes on collected edges of D characterize the lower and upper restrictions in the combination.

**Caputo's** describe sectional Order separation). Caputo's classification is specified by

$$0^{D_t^\gamma} y(t) = \frac{1}{\Gamma(1 - \gamma)} \int_0^t \frac{y^{(m+1)}(\tau)}{(t - \tau)^\gamma} d\tau \quad (4.26)$$

Where  $\alpha = m + \gamma$ ,  $m$  is an integer, and  $0 < \gamma \leq 1$ . Similarly, Caputo's sectional Order addition is well-defined as

$${}^0D_t^\gamma y(t) = \frac{1}{\Gamma(-\gamma)} \int_0^t \frac{y(\tau)}{(t-\tau)^{\gamma+1}} d\tau, \gamma < 0 \quad (4.27)$$

#### 4.3.2. FRACTIONAL ORDER DIFFERENTIATION PROPERTIES

The FOD is succeeding belonging to [34]

The FOD  ${}^0D_t^\alpha f(t)$  concerning of an investigative purpose  $f(t)$  is too systematic

Sectional Order differentiation is correctly matching through integer order one while  $\alpha = n$  is a whole number. Also  ${}^0D_t^\alpha f(t) = f(t)$

The FOD is rectilinear; i.e., aimed at each constant  $a$  and  $b$  takes

$${}^0D_t^\alpha [af(t) + bg(t)] = a {}^0D_t^\alpha f(t) + b {}^0D_t^\alpha g(t)$$

FOD influences accomplish the commutative-law and likewise, please

$${}^0D_t^\alpha [{}^0D_t^\beta f(t)] = {}^0D_t^\beta [{}^0D_t^\alpha f(t)] = {}^0D_t^{\alpha+\beta} f(t)$$

The sectional-Order-differentiation Laplace transforms is well-defined as

$$\mathcal{L}[{}^0D_t^\alpha f(t)] = s^\alpha \mathcal{L}[f(t)] - \sum_{k=1}^{n-1} s^k [{}^0D_t^{\alpha-k-1} f(t)]_{t=0}$$

In specific, condition the byproducts of  $f(t)$  is all identical to 0 at  $t = 0$ , the single consumes

$$\mathcal{L}[{}^0D_t^\alpha f(t)] = s^\alpha \mathcal{L}[f(t)]$$

Presentations of sectional -calculus can remain inventing in the region of regulator classifications. Sectional order calculus decides the derivatives and integrals are of any physical amount. The FOD can be characterized by an all-purpose important operator  ${}^aD_t^\alpha$  as an oversimplification of the differential and integral operators, which is defined as follows

$$a^{D_t^\alpha} = \begin{cases} \frac{d^\alpha}{dt^\alpha} & R(\alpha) > 0 \\ 1 & R(\alpha) = 0 \\ \int_a^t (dt)^{-\alpha} & R(\alpha) < 0 \end{cases} \quad (4.28)$$

Where

a–Lower limit of integration

t–Upper limit of integration

$\alpha$ –Order of fractional differentiation or integration Negative  $\alpha$  - shows integration and positive  $\alpha$  -shows differentiation[35],[36].

- Frequency and time-Domain Investigation of Fractional Order Systems

FOS are straight addition of conventional integer-order schemes. The fractional order system is accepted up-on the FODE, and the FOTF of a particular adjustable scheme can be definite as

$$G(s) = \frac{b_1 s^{\gamma_1} + b_2 s^{\gamma_2} + b_3 s^{\gamma_3} \dots \dots \dots b_m s^{\gamma_m}}{a_1 s^{\eta_1} + a_2 s^{\eta_2} + a_3 s^{\eta_3} \dots \dots \dots a_n s^{\eta_n}} \quad (4.29)$$

Anywhere  $(a_n, b_m) \in \mathbb{R}^2$  &  $(\eta_n, \gamma_m) \in \mathbb{R}^2$

- Frequency Domain Investigation of Fractional Order Systems

We conclude fractional-order basic terms are in three categories: - fractional-order pure differential term, fractional-order derivative term, and fractional-order integral term. It can be understood that, when  $j\omega$  is used to adding for the variable  $S$  in the FOTF model the above, the frequency domain response  $G_a(j\omega)$  can be easily calculated. Thus, Fractional Order Bode diagrams and Nyquist plots can be simply evaluated.

- Investigation Of Fractional-Order Time Domain (IFOTD) Systems

Assessment of Fractional Order time-domain response is additionally difficult. As shown in the succeeding an extraordinary method of a FOD equation.

$$a_1 D_t^{\eta_1} y(t) + a_2 D_t^{\eta_2} y(t) + \dots + a_n - 1 D_t^{\eta_{1n-1}} y(t) + a_n D_t^{\eta_{1n}} y(t) = u(t) \quad (4.30)$$

When  $u(t)$  is characterized by an influenced purpose & its sectional Order derivative. Receive that the  $y(t)$  has zero primary situations. Beginning the Laplace, transform function determine the transfer function

$$G(s) = \frac{1}{a_1 s^{\eta_1} + a_2 s^{\eta_2} + a_3 s^{\eta_3} \dots \dots \dots a_n s^{\eta_n}} \quad (4.31)$$

The fractional-order linear time-invariant (LTI) system can also be represented by the following state-space form

$${}^0D_t^q x(t) = Ax(t) + Bu(t) \quad (4.32)$$

$$y(t) = Cx(t)$$

Where  $x \in R^n$ ,  $u \in R^r$  and  $y \in R^p$  are the state-input and output vectors and  $A \in R^{n \times n}$ ,  $B \in R^{n \times r}$ ,  $C \in R^{p \times n}$  and  $q = [q_1, q_2, q_3 \dots \dots \dots q_n]^T$  is the fractional-order system.

### 4.3.3. STABILITY ANALYSIS OF FRACTIONAL-ORDER SYSTEM

The stability is an exceptionally significant property of the non-linear system that can be examined in several domains' normal conception of BIBO. A fundamental LTI system using impulse response  $h(t)$  toward bounded input bounded output stable if the essential and appropriate condition is fulfilled:-

$$\int_0^{\infty} \|h(\tau)\| d\tau < \infty \quad (4.33)$$

Where the output of the system is defined by convolution

$$f(t) = h(t) * u(t) = \int_0^{\infty} h(\tau) u(t - \tau) d\tau \quad (4.34)$$

The additional very important domain is the frequency-domain. In terms of frequency domain for estimating the stability, transform the  $s$ -coordinate into the complex plane  $G(j\omega)$  and the transformation is understood allowing the transfer-function of the open-loop system  $G(j\omega)$ . Throughout the conversion, all roots of the characteristic equation of a polynomial are a map from  $s$ -coordinate into the critical-point  $(-1, j0)$  in the plane  $G(j\omega)$ . However, we cannot directly use an algebraic tool, for example, Routh-Hurwitz criteria for the fractional-order system, since we don't have a distinctive equation [37]. An interesting point is stable of FOS might root in the right-half of complex  $\omega$ -plane, as presented in fig.4.6 Since the principal sheet of the Riemann surface, is defined  $-\pi$

$\angle \arg(s) < \pi$ , by using the mapping  $w = s^q$ , the corresponding  $w$  domain is defined by  $-\pi < \arg(w) < q\pi$ , and the  $w$  plane region corresponding to the right half-plane of this sheet is defined by  $-q\pi/2 < \arg(w) < q\pi/2$

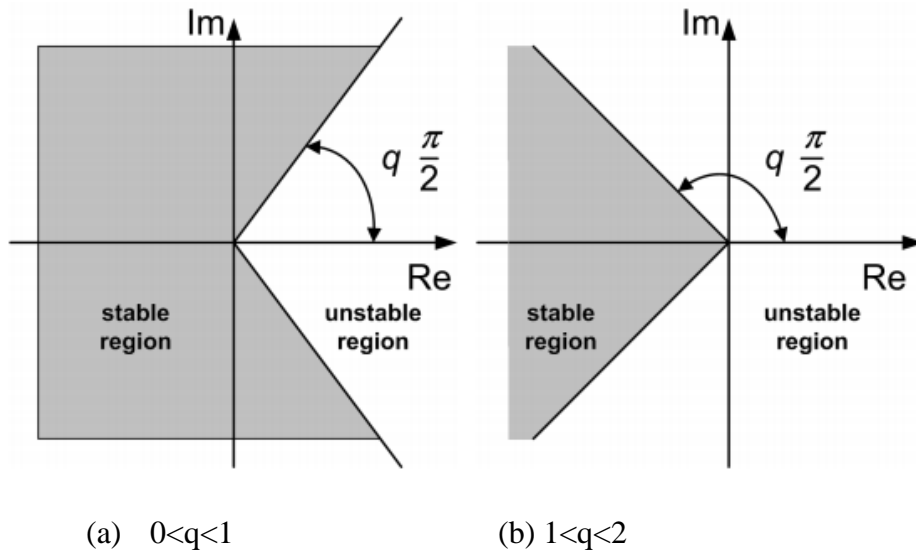


Figure 4.6:- stability region of the fractional-order system

The FO is stable or not if satisfied the following condition

The fractional transfer function of equation (4.29),  $G(s) = \frac{Z(s)}{P(s)}$  is stable if and only if the following condition is satisfied in  $w$ -plane.

$$|\arg(w)| > q\pi/2, \forall w \in \mathbb{C}, P(w) = 0 \quad (4.35)$$

Where  $0 < q < 2$  and  $w = s^q$ . When  $w = 0$  is a particular root of  $p(s)$ , for  $q = 1$ , this is the traditional theorem of pole-place in the complex-plane, no-pole is in the right-plane. The fractional-order linear-time-invariant system state-space model (4.32) is stable if

$$\begin{aligned} & |\arg(\text{eig}(A))| \\ & > \frac{q\pi}{2} \end{aligned} \quad (4.36)$$

Where  $0 < q < 1$  and  $\text{eig}(A)$  represents the eigenvalues of matrix  $A$ .

#### 4.4. TYPES OF FRACTIONAL ORDER CONTROLLER

AFOPID regulator is the extension of the classical PID-controller based on fractional-calculus. For several years, engineering PID controllers have been presenting very commonly used in process industries. The significance of proportional-integral-derivative controls, incessant efforts is existence ended to develop robustness. An automatic regulator

method, FOC is oversimplification integer-order controllers would regulate the robust control achievement and also more accurate.

The FOC which stands the overview of conventional IOC would main to additional detailed & robust governor performance. However, it is practical; the fractional-order simulations involve the FOC toward accomplishing the better assessments. The FOC is functional to rectilinear or non-rectilinear dynamic forces to improved performances of a scheme. Generally, there are four main categories of FOC [38].

- Fractional order PID-controller
- Tilted proportional and integer (TID) controller
- Crone controller
- Fractional lead-lag compensator

#### 4.4.1. $PI^\lambda D^\mu$ CONTROLLER

The FOPID controller has two more adaptable parameters than the PID controller, and the order of the controller can be chosen randomly, so the FOPID controller owns more flexibility. In addition to that, the FOPID controller has great adaptability to the parameter variation of the control system. When the constraint of the control system change contained by a certain range, the system characteristics remain unchanged, so the FOPID controller has the characteristic of strong robustness. Control techniques including feedback control, optimal control, predictive control, neural network control, fuzzy logic control, and so on, have been established meaningfully. However, the PID control technique widely used in several industrial applications such as process control, motor drives, flight control, etc. More than 90% of the control loops in the industry are PID based control loop. This is mostly because of PID controller possesses robust performance to fulfill the international transformation of industry process, simple construction to be easily understood by engineers, and open-mindedness to design and implement. Recently, there are increasing the performance of the proportional-integral-derivative controller by using the knowledge of fractional calculus. Proportional integral derivative controllers are linear and symmetric and they have complications in the occurrence of non-linearity. This problem can be solved by using a fractional-order PID (FOPID) controller.  $API^\lambda D^\mu$  regulator through integrator of actual order  $\lambda$  & differentiator actual order  $\mu$  is a more common cause of the classical PID-controller which delivers more flexibility and robustness in tuning and control by an additional two Degree of Freedom in the system.

These degrees of freedom are related to the orders of the integral and derivative parts which are extended to non-integer (fractional) values. The FOPID control suggested by Podlubny is a simplification of a traditional proportional-integral-derivative regulator expending the fractional multiplication. The TF is well-defined by Equation (4.38).

An F-OPID control is characterized by five constraints, i.e. the Proportional gain ( $K_p$ ) the integral gain ( $K_i$ ), the derivative gain ( $K_d$ ), integral order ( $\lambda$ ), and derivative order ( $\mu$ ) [39]. The FOPID controller is the increase of the classical conventional PID controller based on fractional mathematics. Due to the dominant importance of PID controllers develop their features and robustness [40]:

Fractional-order controllers have received significant responsiveness in the last years both from the academic and industrial point of view. Podlubny suggested an overall form of the IOPID control, is called  $PI^\lambda D^\mu$  control, where the values of  $\lambda$  and  $\mu$  lie between 0 & 1. Related with the PID controller, the FOPID controller has more two adjustable parameters, which makes the parameter tuning more flexible, it is very essential significant for successful the control accuracy, therefore, the  $FOPI^\lambda D^\mu$  a controller applied for managing a Level process in interacting two-tank [41],[42].

The differential equation of fractional order  $PI^\lambda D^\mu$  the controller is shown as equation (3.37)

$$u(t) = k_p e(t) + k_i Dt^{-\lambda} e(t) + K_d Dt^\mu e(t) \quad (4.37)$$

Relating Laplace transforms to this equation with zero initial conditions, the TF of the controller can be expressed by:

$$G_{fc}(s) = \frac{U(s)}{E(s)} = K_p + K_i s^{-\lambda} + K_d s^\mu, \quad (\lambda > 0, \mu > 0) \quad (4.38)$$

Taking  $\lambda = 1$  and  $\mu = 1$  it is the conventional PID controller, if  $\mu = 0$  and  $\lambda = 1$  it is the conventional PI controller, if  $\lambda = 0$  and  $\mu = 1$  it is the unadventurous PD controller and if both  $\mu \& \lambda = 0$ , it  $PI^\lambda D^\mu$  can be seen that the adjustable range of the fractional  $PI^\lambda D^\mu$  the controller is wider than the conventional PID controller, therefore, the guideline

performance of the FOC is greater than the IOPID controls.

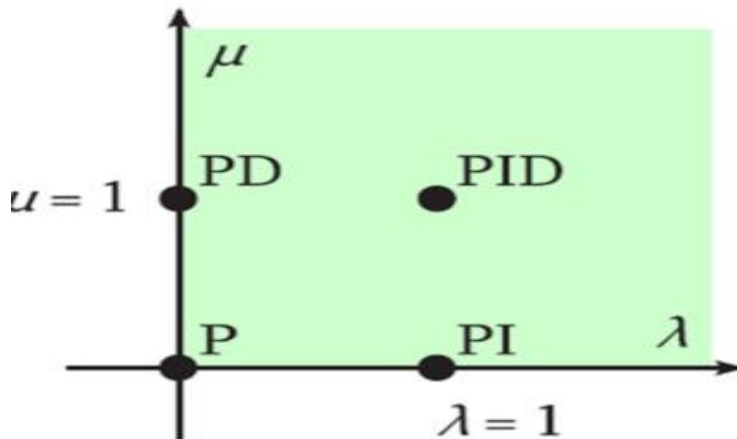


Figure 4.7:- Graphs of PID controller

#### 4.4.2. TILTED PROPORTIONAL AND INTEGER (TID) CONTROLLER

Tilted-Proportional and Integral (TID) Controller is to provide an enhanced feed-back loop controller gains of the classical Proportional integral controller. In TID structure the proportional compensating unit is exchanged with a compensator having a transfer function characterized by  $1/s^{1/n}$  or  $1/s^{-1/n}$ . This compensator is herein referred to by way of a ‘‘Tilt’’ compensator, as it delivers a feedback gain as a function of frequency which is tilted or formed for the gain/frequency of a conservative or positional recompense unit. The perfect controller is herein referred to as a Tilt-Integral Derivative (TID) controller. It contains three components tunable feedback loop control system which contains a proportional integral derivative controller. The individual transformation from the conventional controller is that the proportional recompensing part of the system is replaced with a more suitable compensator which is having a transfer function. The term ‘Tilt’ suggests that it can provide a feed-back improvement as a frequency purpose that is shaped or tilted to improvement frequency of predictable reparation entity. A transfer function of TID can be written:-

$$G(s) = K_t(1/s)^{\frac{1}{n}} + \frac{k_i}{s} + sk_d \quad (4.39)$$

Where n is a non-zero real number, the above transfer function is shown in the figure below as:

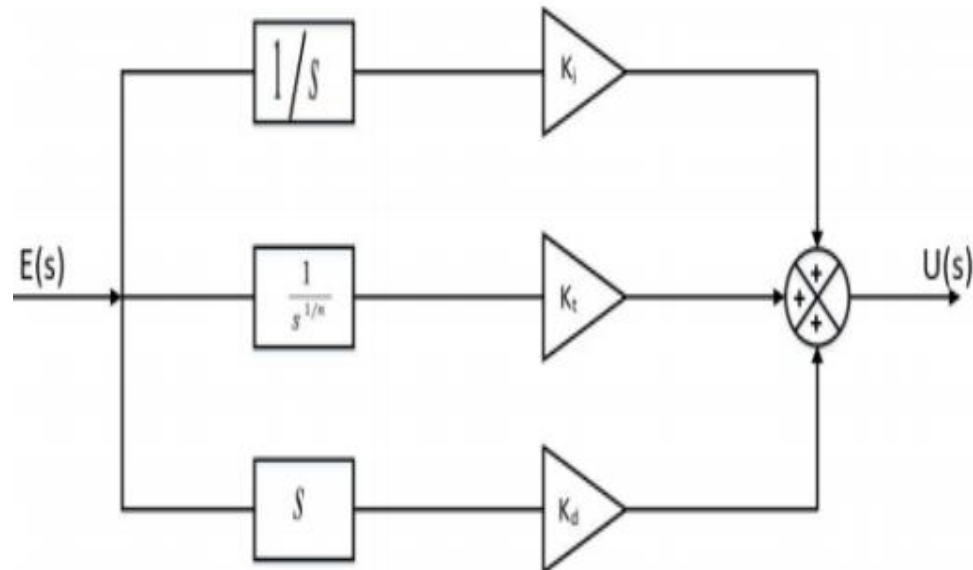


Figure 4.8:- Tilt-integral derivative controller

The effects of TID controller can be summarized as:

- ✓ Simple Tuning
- ✓ Feedback control is improved
- ✓ Disturbance rejection ratio is improved
- ✓ system parameter variation has less influence on closed-loop response

The main parameter for tuning in the TID controller is the tunable coefficient “n” and the other parameters ( $K_i$  and  $K_d$ ) are the same as we used in the conventional PID controller. Thus, the most optimum value of “n” is obtained by this technique for the lower percent of under/overshoot, settling time, and steady-state error[43].

#### 4.4.3. CRONE CONTROLLER

Crone Controller is a French acronym for fractional-order robust control. By the use of these controllers, it is conceivable to confirm almost constant closed-loop features and ensure a small variation of the CL system stability degree in the meanness of the plant perturbation and improbability in model parameters. A crone controller exists in three generations. Once the phase and the margin are constant, the first generation is used. Though, when the phase is only constant nearby the unity gain frequency, the second generation is used. The structure's instruction for the first two generations is non-integer changing between 0 and 1 however, for the third generation, the complex order originates. It has a frequency domain design methodology engaging fractional differentiation. It is

possible to control plants, unstable, time-varying, and non-linear plants with crone controller. one of the applications of crone controllers has been in the domain of flexible transmission, car suspension control, and hydraulic actuator.

The Crone controllers are designed using this method. To move from the rational or fractional representation of the rational form transfer function, Oustaloup method is used.

$$\text{Fractional form: } C(s) = \frac{K_0}{s} \left( \frac{1 + \frac{1}{\omega_n}}{1 + \frac{1}{\omega_b}} \right)^n \quad (4.40)$$

Rational form:

$$I_N^n(s) = \frac{K_0}{s} \prod_{i=1}^N \left[ \frac{1 + \frac{s}{\omega_i}}{1 + \frac{s}{\omega_u}} \right] \quad (4.41)$$

Where  $N$  characterizes the estimate of cells (commonly essentials one cell which is collected of a pole and zero with a minimum of four cells to reconstruct the fractional system),  $k_0$  is the gain responsible to set the unit gain frequency  $\omega_u$ ,  $\omega_i$  and  $\omega_u$  being respectively the poles and the zeros of the rational transfer function. Regarding the combination of the crone controller, the identical method is used for the three generations. The posterior method is organized where the TF of the controller is deduced after defining the behavior of the open-loop system. Hence, the transfer function of the open-loop is the following:

$$\beta(s) = \beta_0 \left( \frac{1 + s/\omega_b}{s/\omega_b} \right)^{n_b} \left( \frac{1 + s/\omega_h}{1 + s/\omega_b} \right)^n \left( 1 + \frac{s}{\omega_h} \right)^{-n_h} \quad (4.42)$$

- The first step contains defining the necessary specifications for the synthesis of the nominal plant transfer function. The frequency method is also used in this part as the standardization of the sensitivity functions is easier.
- The second step consists of reselecting the frequency closed-loop conditions into open-loop frequency specifications for the nominal plant. These new conditions take into explanation the plant behavior at:
  - ✓ Low-frequencies require respectable accuracy in the steady-state;
  - ✓ intermediate frequencies, specifically around the frequency  $\omega_u$ , to get the stability degree robustness;
  - ✓ High frequencies to have good input plant sensitivity. Hence,  $\omega_b$  and  $\omega_h$  characterize the low as well as high intermediate frequencies,  $n$  is the

fractional-order fluctuating among 1 and 2,  $n_b$  and  $n_h$  is asymptotic order performances for small and high frequency and  $\beta_0$  is a constant that guarantees a unit gain at the frequency  $\omega_u$  [44].

#### 4.4.4. FRACTIONAL LEAD-LAG COMPENSATOR

Fractional Lead-Lag compensators are commonly used to stabilize slightly stable systems. Lag compensators are mostly used to decrease the amplitude of the high-frequency loop gain of a system. The use of fractional order elements in this lag-lead compensator gives greater flexibility to fashionable, to form the loop frequency responses since the order of the clean can take any real value as a replacement for of only integer values the TF of a generic fractional-order lead-lag compensator [45].

The fractional lead-lag controllers are often desired in the strategy of control systems. Although the fractional-order lead-lag controllers are comparable to other terms of the control structure, they are different in terms of placement of the zeros and poles.

$$C(s) = K_c \left( \frac{s + \frac{1}{\lambda}}{s + \frac{1}{x\lambda}} \right)^\alpha = K_c x^\alpha \left( \frac{\lambda s + 1}{x\lambda s + 1} \right)^\alpha, 0 < x < 1 \quad (4.43)$$

Where  $0 \leq \alpha \leq 2$ , the transmittance corresponds to the frequency bounded fractional derivative/integrator which is at the very origin of the crone control.

#### 4.5. MATLAB TOOLBOXES FOR ANALYZING FRACTIONAL ORDER SYSTEMS

In recent years, as fractional calculus develops more and more broadly used across different academic disciplines, there are growing difficulties for the numerical tools for the calculation of fractional integration/differentiation or the simulation of fractional-order systems.

##### A) @fotf

@fotf (fractional-order transfer- function) is a control tool-box for fractional-order systems developed by **Xue**. Most of the FOTF exclusive are extended from the MATLAB builtin-functions. @fotf tool-box uses the overload programming method to enable the related methods of the Matlab built-in functions to covenant through fractional-order models. The FOTF objects created from it can be collaborating utilizing generated from the MATLAB

transfer-function course. Conversely, the over-loading of related functions such as impulse (), step (), etc. @fotf toolbox supports the time delay in the transfer function. It doesn't straight support the TFmatrix; hence, multi-input multi-output systems can't be simulated. Nevertheless, meanwhile, it delivers Simulink-block -encapsulation of the complicated function @fotf (), MIMO correlation recognized via physically adding round interfaces Simulink – block – diagrams. A draw-back of the fraction order transfer function is that the sample time has a moderately big influence on accurateness [46].

#### B) Ninteger

Ninteger, a non-integer MATLAB control toolbox, is a toolbox proposed to help with emerging FOC and evaluating their performance. Fractional order Simulink block diagram are involving the following function, such as nid, nipid, fotf, etc. [47]

#### C) CRONE

The crone-Tool-box, established meanwhile 19<sup>th</sup> by acrone group, is a Matlab and Simulink Toolbox devoted to presentations of non-integer derivatives in industrial skill [48]. It developed beginning the inventive script version of the remaining programming. A respectable article of the crone toolbox are certain of the approaches is appreciated for multi-input multi-output fractional transfer functions. Numerous extra toolboxes are encouraged by crone, e.g. ninteger and FOMCON. The draw-back of the crone toolbox are intervals suspension can't be combined interested in the engendered FOTF. Crone a MATLAB tool-box considerably more dominant than just simulating FOS.

#### D) FOMCON

The FOMCON (Fractional- Order-Modeling, and Control) *toolbox* is established by Aleksei Teplov [16]. Its kernel work systems in @fotf, ninteger, and Crone. Summarizes the particular main function of individuals three-fractional-order (FOMCON) toolboxes, the relative FOMCON within three-toolboxes is obtainable in fig 4.9 particular distinguished deviations of the unique @fotf are

- Newfotf() practices string parser to allow operators to input-transfer-function as a string

- tf2ss() are loaded and foss() are additional, which varieties the adaptation between fof(0entity and foss()article. The croneMATLAB-toolbox is consistently intelligent to do atask;contrariwise, an index is encoded in Matlab P-code format. In this project,the FOMCON Matlab toolbox is used because it contains the main functionalities of fof, ninteger, and Crone toolboxes[49].

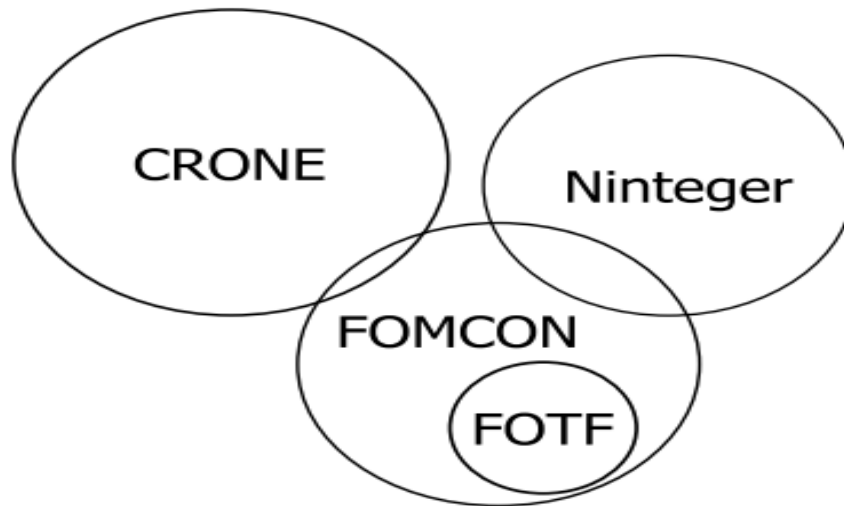


Figure 4.9:- FOMCON relation to other fractional-order MATLAB toolboxes

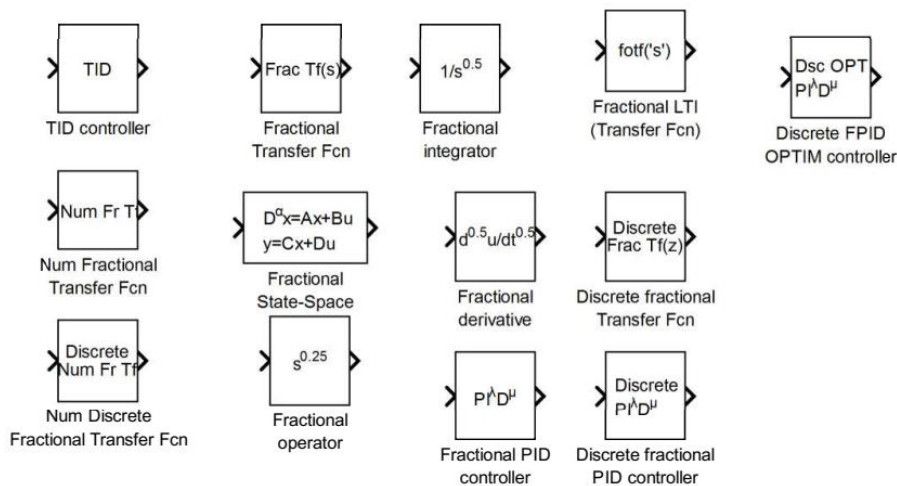


Figure 4.10:- FOMCON Simulink library

#### 4.6. INTEGER ORDER VS FRACTIONAL ORDER PID CONTROLLER

Even though the integer-order proportional integral derivative control mechanism well concerning the consummation of the design description. Nevertheless, through the modification of constraint assessment & aging, the yield is likely to denigrate and change. A conventional proportional-integral-derivative Controller doesn't require the

competent toward accomplishing within a changing constraint. By this yield incline toward

differing after its insignificant importance FOPID the constraint of a traditional PID regulator can be overawed by the impression of a additional modification constraint in order of integrator & differentiator .

Surely, moreover situation proportional gain  $K_p$ , integral gain  $K_i$ , and derivative, gain  $K_d$  have two additional constraints  $\lambda$  and  $\mu$ , whatever develops the opportunity of control scheme design. The FOCoversimplifies the IO PID design and rises from point to surface. The development improves additional suppleness to CSD and we can control our real-world method more effortlessly. Within the benefit of FOPID controls, it is conceivable to design a controller to guarantee that the CL scheme is robust to improvement variance & step reply shows a stage bode scheme of the structure is horizontal achievement crossover frequency as a consequence technique develops added robustness to improvement modification.  $PI^\lambda D^\mu$  Controllers are a smaller amount of sensitivity to constraint variation. All furthermost of the practical progressions can be competently demonstrated within the help of FO difference equivalences moderately than integer ones [40]. Consequently, FOC is appropriately aimed at these sectional-order models to represent better control.

#### 4.6.1. ADVANTAGE OF FRACTIONAL AND INTEGER ORDER PID CONTROLLER

Through related an integer-order controller, & fractional-order is theoretical to suggest the following recompenses [50]

- ✓ If the constraint of measured variable deviations, sectional-order PID controller are less sensitive rather than traditional PID controller
- ✓ FOPID controller has established significant consideration in the preceding years together after the theoretical & manufacturing themes of view concerning the ordinary PID controllers as five parameters to choose.

Generally, the use of fractional order models and controllers is expected to lead a momentous overall improvement of industrial control loop quality thus providing an increase in control system precision, performance, and energy efficiency. When FOPID

controllers implementation for real-time used estimate method because two additional parameters of the FOPID controller to improve the flexibility of the system.

The extension of derivation order from integer to fractional numbers provides a more flexible tuning strategy and an easier achieving of control requirements concerning classical controllers. The potential use of FOPID controllers lies in the industrial domain, where the majority of control loops are based on traditional PID controllers, including those dedicated to liquid level control. The application of FOPID controllers to the problem of liquid level control is justified since such controllers offer more tuning flexibility parameter and allow taking into account more robustness criteria and also unmolded dynamics of real industrial systems, small advantages in performance arising from using fractional-order controllers (FOPID) in place of integer order of Proportional Integral Derivative (IOPID) will lead to an overall enhancement [51]. The most common design method for fractional-order controllers is based on frequency domain analysis [52].

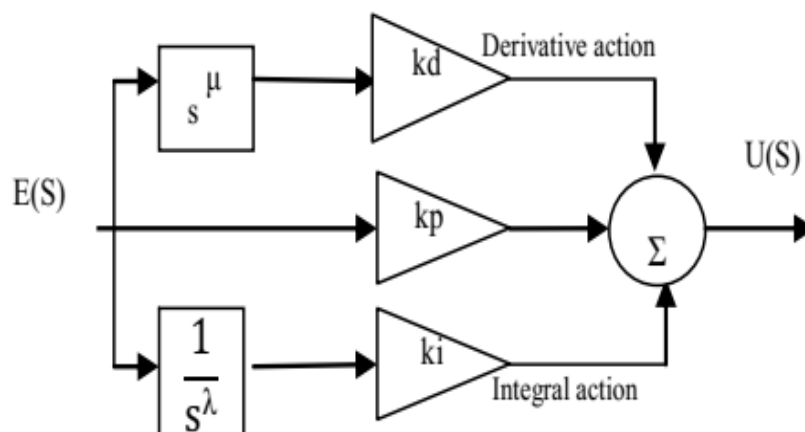


Figure 4.11:- Block diagram of FOPID controller

It can be predicted that the  $PI^\lambda D^\mu$  controller may upgrade the efficiency of the system. Additional feature lies in the fact that  $PI^\lambda D^\mu$  controllers are low sensitive to the constraints variations of the controlled system also FOPID provides further flexibility in the controller compared with the conventional PID controller [53]. The feedback control loop of a fractional-order system with a fractional controller is similar to the integer-order the feedback control loop is shown in fig 4.11

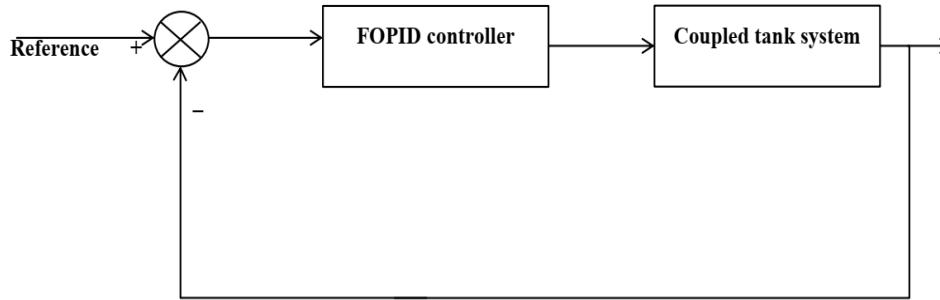


Figure 4.12:-Block diagram of the coupled tank system with FOPID controller

#### 4.7. INTRODUCTION TO LINEAR QUADRATIC REGULATOR

A principle of optimal techniques is working dynamics scheme at minimum cost function of a method whose subtleties are specified through the convenience of rectilinear difference equivalences & cost function through the rectilinear quadratic problematic. The situation of a regulator that manages whichever an apparatus or method is found by an exact process that reduces a cost function contains considering features. A truthful procedure is an impartial purpose that essential to be reduced in the strategy practice. Fashionable, two methods controls are planned & clarified in detail which is the FOPID & LQR controller. Furthermore, the succeeding design condition has completed evaluating the performance of LQR control schemes those performances are evaluated by transient response. To stunned selected problems faced by the PID regulator, and develop another controller method are linear-quadratic-regulator. The performance measure of the linear-quadratic-function collected of the state space form & control signal.

Linear quadratic regulator [54],[55] strategy system is well recognized in modern optimal control theory and has been commonly used in many applications. Thus, the LQR theory has established significant attention since the 1950s. The LQR method seeks to find the optimal controller that reduces a given cost-function. Two parameters of cost function matrices are Q and R, which consider the state-vector and the system-input respectively. These allowance matrices regulate the consequences on the excursion of state-variables and control signals. One practical method is to Q and R to be a diagonal matrix. The value of the parameter in Q and R is related to the controlled plant. The LQR is a control design that delivers the best conceivable assessments regarding particular portion of enactment. Linear quadratic regulator is overcome the strategy of state – feedback controller K such that the

opportunity of cost-function  $J$  is reduced. In this thesis work, the function of state feedback control is used to control strength, increase the amplitude of the response with a desired point and gives a better response of system. The commonly LQR used for analysis the state feedback gain from closed loop system. Linear Quadratic Regulator is, in fact, a feedback controller for control the level of the process that provides the solution in the cases wherever the system dynamics are described by a set of linear-differential-equations and the cost is designated by a linear-quadratic quadratic function. The LQR procedure is an automated way of outcome a raised feedback controller.

Control engineers can also prefer alternative ways for designing the appropriate controller like a full-state feedback controller or known as pole-placement, LQR has proposed to regulator the Level in a coupled-tank arranged in the form of the interacting system. Where the parameters of controller governing the system are established via expending analytical modeling to reduce the cost function within weighting structures carried by the engineer in Layman's terms. The LQR modeling is the core, just a mechanized technique of finding a suitable state feed-back controller description of the LQR system [56].

#### 4.7.1. LQR CONTROLLER DESIGN

The LQR is an optimal technique in modern control theory that uses a state-space method to investigate a system. Using state-space approaches it is comparatively simple to effort utilizing a multi-output system [57]. The LQ controller designed is categorized as optimal control techniques. The purpose of the LQR is to realize a system with practical components that will provide the desired set point. The preferred point can be readily stated in terms of time-domain performance indices. For example, the maximum overshoot and rise time for a step input are respected time-domain indices, In the case of steady-state and transient performance, the performance indices are normally specified in the time -domain [58], [59], [60]. The schematic of this type of control systems are existing in Fig. 4.13.

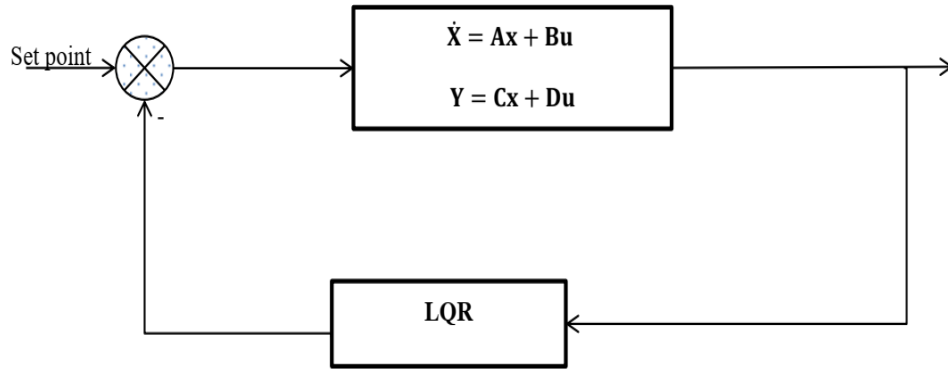


Figure 4.13:- Block diagram of the coupled – tank using LQR optimal controller

#### 4.7.2. PRINCIPLES OF LQ PERFORMANCE ANALYSIS

Optimal Techniques say (FSB) achievement matrix “K” is considered&also invented that completely  $n$  state variable are computable ones and obtainable for the regulator[61], [62]. the state and output equivalence can be specified the fig. (4.12)

$$\dot{x} = Ax + BU, y = Cx + DU \quad (4.44)$$

Where  $x$  variable is a column vector of length  $n$ ,  $u$  is the control input – vector of length  $r$ ,  $A$  is an  $(n \times n)$  square state matrix;  $B$  is an  $(n \times r)$  input matrix;  $y$  is a column output vector,  $C$  is an  $(m \times n)$  the output matrix and lastly,  $D$  is an  $(m \times r)$  through response matrix.

The LQR is used to control complex-systems that require a high-performance. a system described by linear differential equations can be shown in the steady-state form given in equation (4.1).

For a continuous – time described by

$$\left. \begin{aligned} \dot{x} &= Ax + BU \\ y &= Cx + DU \end{aligned} \right\} \quad (4.45)$$

By a cost function well-defined as

$$J = \int_0^{\infty} (X^T Q x + U^T R U) dt \quad (4.46)$$

Where  $Q$  and  $R$  are the weight matrices,  $Q$  – is required to be a positive-definite or positive-semi-definite symmetry matrix;  $R$  – is required to be a positive definite symmetry matrix. One practical method is to  $Q$  and  $R$  to be a diagonal matrix. The value matrix  $Q$  and  $R$  are related to its involvement in the cost function  $J$ . The feedback control

law that reduces the value of the cost. Designing a linear quadratic regulator controller consist of the following steps

Step 1: Q and R matrix, minimizing J must be chosen.

Step 3: Optimum feedback gain matrix K is calculated using (4.48).

Step 4: System response is checked. If the system response is not met with the required specifications, repeat all steps[63].

$$U = -Kx \quad (4.47)$$

K – is given by

$$K = R^{-1}B^T P \quad (4.48)$$

P can be initiated through resolving the nonstop time (ARE)[64]

$$A^T P + PA - PBR^{-1}B^T P + Q = 0 \quad (4.49)$$

#### 4.7.3. ALGEBRAIC APPROACH FOR A SECOND-ORDER SYSTEM

Contemplate a second-order LTI scheme characterized in well-regulated canonical form,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ A_{21} & A_{22} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ B_{21} \end{bmatrix} u \quad (4.50)$$

Scheming (FSB) regulator through linear quadratic regulator (LQR) needs the minimization of a cost function specified in eqn(4.45) which seats the weight not single on mechanism input but likewise on the state of the structure. Hence the (SFB) law is specified by equation (4.46). One of the important structures LQR- are the Q – should be asymmetric positive semi-definite matrix and R must be a positive-definite-matrix. So, Q and R are weighting matrices, are chosen as

$$Q = \begin{bmatrix} q_{11} & 0 \\ 0 & q_{22} \end{bmatrix}, R = r, p = \begin{bmatrix} p_{11} & p_{12} \\ p_{21} & p_{22} \end{bmatrix} \quad (4.51)$$

In an optimal method, the (SFG) matrix can be calculated

$$K = R^{-1}B^T P = \frac{1}{r} [0 \quad B_{21}] \begin{bmatrix} p_{11} & p_{12} \\ p_{21} & p_{22} \end{bmatrix} \quad (4.52)$$

K

$$= \frac{B_{21}}{r} [p_{21} \quad p_{22}] \quad (4.53)$$

The element of p matrix analysis through (ARE) (4.28).

$$\begin{bmatrix} 0 & A_{21} \\ 1 & A_{22} \end{bmatrix} \begin{bmatrix} P_{11} & P_{12} \\ P_{21} & P_{22} \end{bmatrix} + \begin{bmatrix} P_{11} & P_{12} \\ P_{21} & P_{22} \end{bmatrix} \begin{bmatrix} 0 & A_{21} \\ 1 & A_{22} \end{bmatrix} - \frac{1}{r} \begin{bmatrix} P_{11} & P_{12} \\ P_{21} & P_{22} \end{bmatrix} \begin{bmatrix} 0 \\ B_{21} \end{bmatrix} \begin{bmatrix} 0 & B_{21} \end{bmatrix} \begin{bmatrix} P_{11} & P_{12} \\ P_{21} & P_{22} \end{bmatrix} + \begin{bmatrix} q_{11} & 0 \\ 0 & q_{22} \end{bmatrix} = 0 \quad (4.54)$$

$$\begin{bmatrix} P_{21}A_{21} & P_{22}A_{21} \\ P_{11} + P_{21}A_{22} & P_{12} + P_{22}A_{22} \end{bmatrix} + \begin{bmatrix} P_{21}A_{21} & P_{11} + P_{21}A_{22} \\ P_{22}A_{21} & P_{21} + P_{22}A_{22} \end{bmatrix} - \frac{1}{r} \begin{bmatrix} P_{12}P_{21}B_{21}^2 & P_{12}P_{22}B_{21}^2 \\ P_{22}P_{21}B_{21}^2 & P_{22}^2B_{21}^2 \end{bmatrix} + \begin{bmatrix} q_{11} & 0 \\ 0 & q_{22} \end{bmatrix} = 0 \quad (4.55)$$

$$\begin{bmatrix} 2P_{21}A_{21} - \frac{P_{12}P_{21}B_{21}^2}{r} + q_{11} & P_{22}A_{21} + P_{11} + P_{21}A_{22} - \frac{P_{12}P_{22}B_{21}^2}{r} \\ P_{11} + P_{21}A_{22} + P_{22}A_{21} - \frac{P_{22}P_{21}B_{21}^2}{r} & 2(P_{12} + P_{22}A_{22}) - \frac{P_{22}^2B_{21}^2}{r} + q_{22} \end{bmatrix} = 0 \quad (4.56)$$

#### 4.7.4. THE CLOSED LOOP SYSTEM

The CL state equation of the system toinscribed as

$$\dot{x} = (A - BK)x(t) = (A - BR^{-1}B^TP)x(t) \quad (4.57)$$

Eigen-values of a CL methodessentialhave negatives real-parts of the closed-loop method stable. According to a direct-substitution technique fordesign pole-placement control.

$$|SI - A + BK| = 0 \quad (4.58)$$

Replacement ofequivalent method matrix A&Binput the (SFC) gain matrix-K in a characteristic

$$S^2 + \left( \frac{P_{22}B_{21}^2}{r} - A_{22} \right) S + \left( \frac{P_{12}B_{21}^2}{r} - A_{21} \right) = 0 \quad (4.59)$$

The general form of second-order characterized equation forms

$$S^2 + 2\xi\omega_n s + \omega_n^2 = 0 \quad (4.60)$$

Comparing Equations (4.59) and (4.60), the expressions for  $p_{21}$  and  $p_{22}$  can be obtained as given below

$$\left. \begin{aligned} p_{12} &= \frac{r(\omega_n^2 + A_{21})}{B_{21}^2} \\ p_{22} &= \frac{r(2\xi\omega_n + A_{22})}{B_{21}^2} \end{aligned} \right\} \quad (4.61)$$

Or In general, for a second-order system, when  $Q$  is a diagonal matrix and  $R$  is a scalar quantity, the elements of the Riccati matrix  $P$  can be calculated:

#### 4.7.5. DESIGN PROCEDURE

The numerical design of a plant is organized in a state-space differential form.

- Identify the essential damping-ratio ( $\xi$ ) & accepted frequency ( $\omega_n$ ) a pla.
- Achieve the SF) matrix  $K$  in terms of the transformation-matrix  $P$ ,
- Decide the actual characteristic equivalence of the plant.
- Additional transformation matrix ( $P$ ) is (ARE) & achieve the terminologies.
- Adjust the value of  $R$  &  $Q$  matrix.
- Determine the gain matrix of the optimal techniques.
- Find out the system confirms that the strategy happens (sp) in terms of transient response.

#### 4.7.6. DYNAMIC BEHAVIOR OF THE PLANT

To understand and control complex systems, we must obtain their quantitative mathematical models by analyzing the relationships between the variables of the system and employing Laplace transform or other mathematical tools [17]. A set of linear – differential-equation remained formulated to obtain a model transfer function that described the physical system of the plant.

#### 4.7.7. PERFORMANCE MEASURES CRITERIA

- To test our model, we choose to use performance measures. These measures will be used in analyzing the performances of FOC and LQR.

Transient-response: - one of the furthermost essential characteristics of control is the reply of a system as a function of time .it can be defined in terms of two issues.

The speed of the response as represented by the rise-time ( $T_r$ ) and The closeness of the response to the desired response as represented by the overshoot ( $O_s$ ) and settling-time ( $T_s$ )[65],[66]

#### 4.8.LINEAR QUADRATIC REGULATOR DESIGN FOR COUPLED Tank

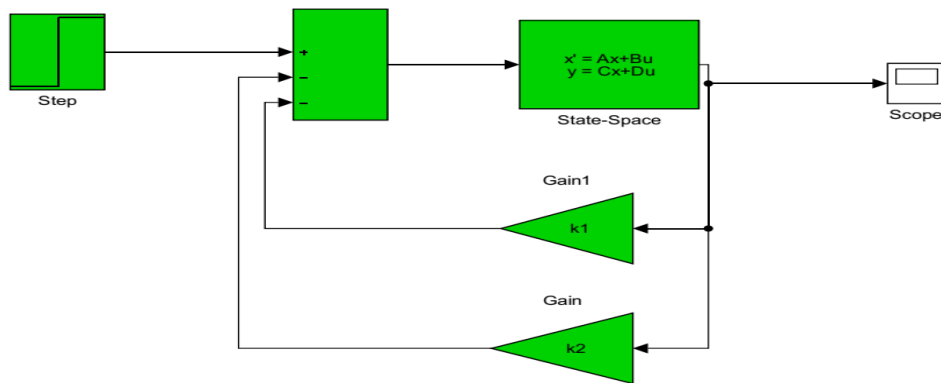


Figure 4.14:- LQR controller design for the interacting system

It is important to realize the optimal control approach to feedback design is very different from classical control. Linear quadratic regulator, it is characterized by a reference input approach to zeros.

#### 4.9.DESIGN LINEAR QUADRATIC REGULATOR WITH INTEGRAL ACTION FOR CONTROLS COUPLED TANK SYSTEM

From a linear scheme in the form of State modeling are:-

$$\left. \begin{aligned} \dot{x}(t) &= Ax(t) + Bu(t) \\ y(t) &= Cx(t) + Du(t) \end{aligned} \right\} \quad (4.69)$$

The control strategy using state feedback has been applied to allocate the pole of the closed loop system (if the system is a completely controllable state) in any position, chosen to meet design specification. An advantage of LQR method when compared to the allocation method is that the first one provides a systematic mode of calculation for state feedback control gain matrix.

The quadratic optimal regulator aims is to find matrix  $k$  for the optimal control vector gain by so that the cost function in is minimized equation (4.47).

Determining the parameter in the above equation (4.44) must be design the LQR controllers. Analyzing a coupled tank system model, it can be seen verified that the output of a process level dynamics behavior of the coupled tank shown figure 3.2. Therefore it is necessary to place an integrator into controller in order to eliminate the static error between tracking reference and controlled variable. in other word; the LQR control must have integral action. Block diagram of the LQR with integral action shown in figure and named specifically as LQI.

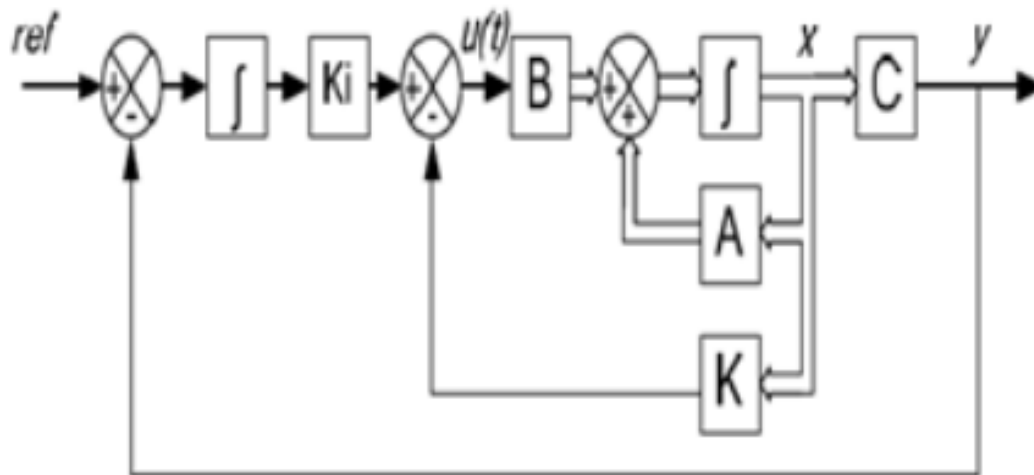


Figure 4.15:- block diagram of LQI of state feedback control systems

The LQR with integral action (LQI) assumes that the following expanded matrix arrangement equation (4.69):-

$$A_{new} = \begin{bmatrix} A & 0 \\ -C & 0 \end{bmatrix} \quad (4.70)$$

$$B_{new} = \begin{bmatrix} B \\ 0 \end{bmatrix} \quad (4.71)$$

$$C_{new} = [c \ 0] \quad (4.72)$$

$$D = 0$$

Whose gain is given by:-

$$K = [k_1 k_2 k_3 \dots \dots k_{n+1} \ -k_i] \quad (4.73)$$

$k_{n+1}$  – state feedback gain

$k_i$  – integral action error gain

In this case, the model cost is defined by equation (4.46)

The gain  $k$  is obtained solving Riccati equation for  $p$  using equation (4.49)

After that obtaining  $p, k$  can be obtained equation (4.48). The physical implementation this control system, every state variable must be measurable and variable for feedback. The optimal gain of the state feedback vector is determined using *matlab* command  $[k, e, p] = \text{lqr}(A_{new}, B_{new}, q, r)$ . This command determine the optimal gain of feedback matrix  $k$ .

FRACTIONAL ORDER PID DESIGNED FOR COUPLED TANK SYSTEM

An open-loop transfer-function of interacting coupled tank systems as derived in equation (4.86) is given. A simulated model for the fractional-order PID control system is as shown in figure 4.15. The difference of (DSP) & the (MV) goes to aFOC. FOPID controller combines PID controllers gain and in addition to two parameters those are lambda ( $\lambda$ ) with integral gain and Miu ( $\mu$ ) with derivative gain

$$G(S) = \frac{R_2}{\tau_1\tau_2S^2 + (\tau_1 + \tau_2 + A_1R_2)S + 1} \tag{4.74}$$

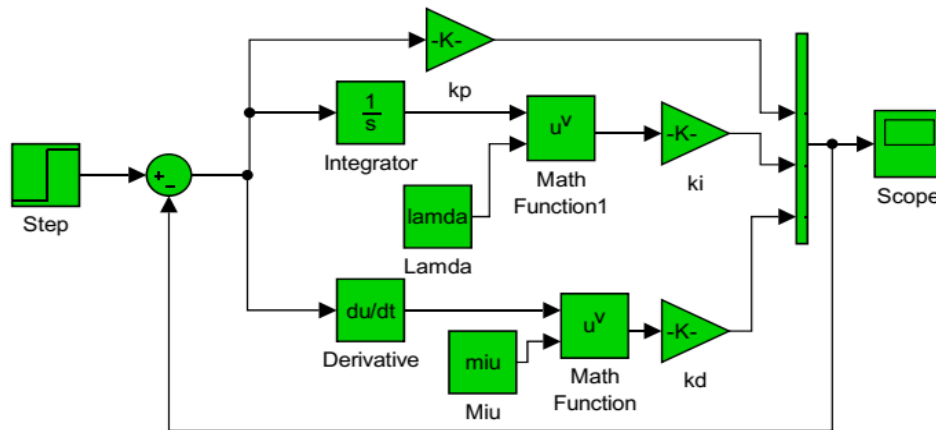


Figure 4.16: Interacting system control using FOPID controller

4.10. PARAMETERS OF LEVEL CONTROL SYSTEM

The achievement of a process-level control system in interacting coupled tank for and compare the reply of the system with fractional order controller and linear quadratic regulator (LQR), the selected parameter of the level tank is specified in table 4.6.

No	Description	Parameter	value	Units
1	Area of tank1	$A_1$	250	$\text{cm}^2$
2	Area of tank2	$A_2$	250	$\text{cm}^2$
3	Resistance of tank1	$R_1$	0.01	$\text{sec./cm}^2$
4	Resistance of tank2	$R_2$	0.01	$\text{sec./cm}^2$
5	Height of tank1	$H_1$	30	Cm
6	Height of tank2	$H_2$	15	Cm

Table 4.6:- parameter of level control systems

Since the open-loop transfer function of the system is given above by equation (4.74) and with equation (4.10) and the unknown open-loop system parameters,  $\xi$ , and  $\omega_n$  are obtained. Using the parameters given in table 4.6.

$$k = \frac{R_2}{\tau_1 * \tau_2}$$

$$\left. \begin{aligned} \omega_n &= \sqrt{\frac{1}{\tau_1 * \tau_2}} \\ \xi &= \frac{\tau_1 + \tau_2 + R_2 A_1}{2\omega_n} \end{aligned} \right\}$$

For interacting system coupled tank  $K$ ,  $\xi$  and  $\omega_n$  can be obtained by the above method. Therefore, the open-loop transfer function of the interacting coupled system without disturbance parameter can be calculated as follows.

$$G(S) = \frac{0.0016}{s^2 + 1.2s + 0.16} \quad (4.75)$$

The response of the open-loop process without the controller shown below

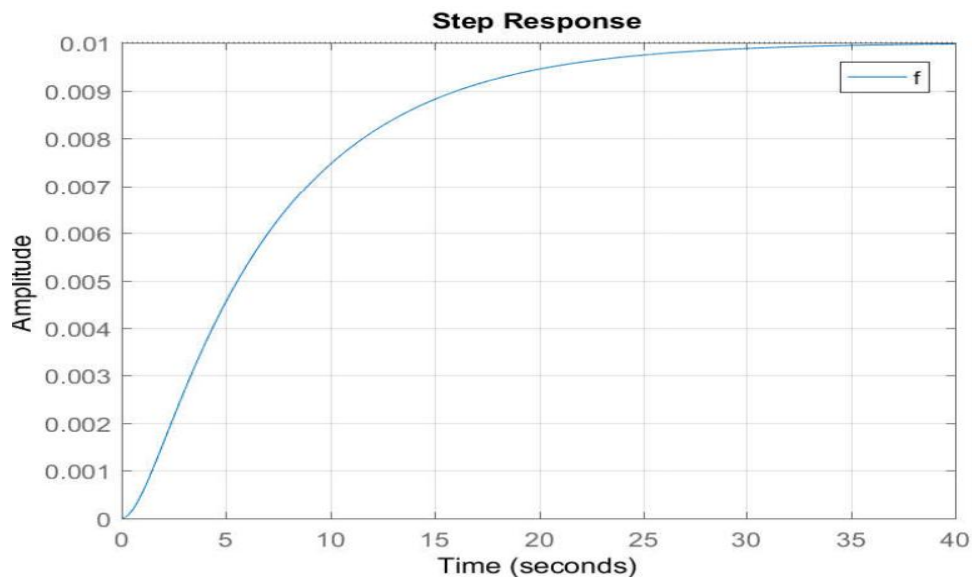


Figure 4.17:- Response of the open-loop system

When the disturbance added to the system, the disturbance transfer function can be

$$G_d(s) = \frac{H_2(s)}{D_{in}(s)} = \frac{0.01}{2.5s + 1} \quad (4.76)$$

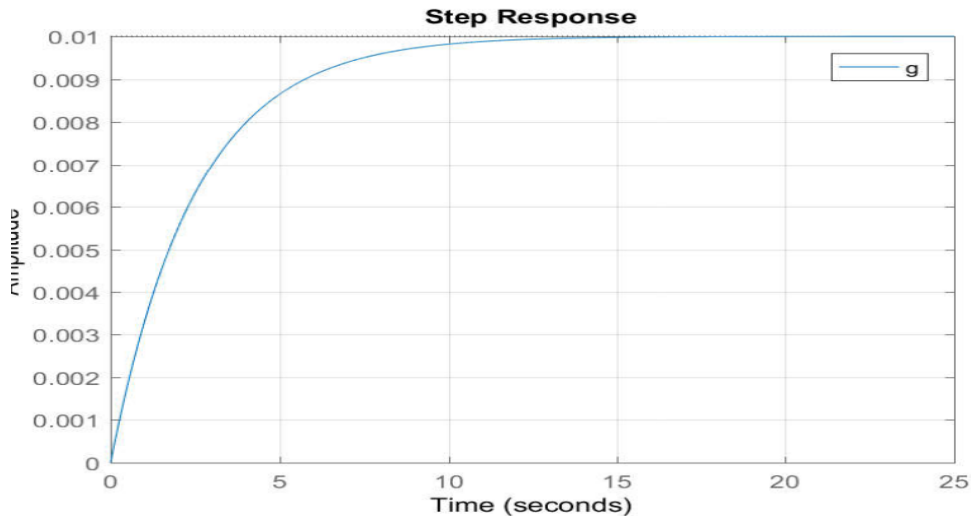


Figure 4.18:- Output response of the open-loop system when disturbance added

The selecting the desired parameters of a closed system and using the relation in equation (4.22), the gain of the classical Proportional –integral-derivative controller gain is considered which produce particular pole placement at specific-damping and frequency, providing relative dominance ( $\alpha$ ) in selected iteratively by examining the accurateness of system response. Here specific considerations of closed loop scheme preferred are  $\xi^{cl} = 0.95$ ,  $\omega_n^{cl} = 2 \frac{rad}{sec}$  selecting the appropriate value of the relative dominance by trial and error by checking the accuracy of system response, the closed-loop transfer function of fractional order Proportional Integral Derivative (FOPID) when the value of  $\mu = \lambda = 0.5$ .

$$G(s)_{cl} = \frac{4.5s + 11.136s^{0.5} + 7.6}{s^{2.5} + 1.2s^{1.5} + 4.5s + 12.136s^{0.5} + 7.6} \quad (4.77)$$

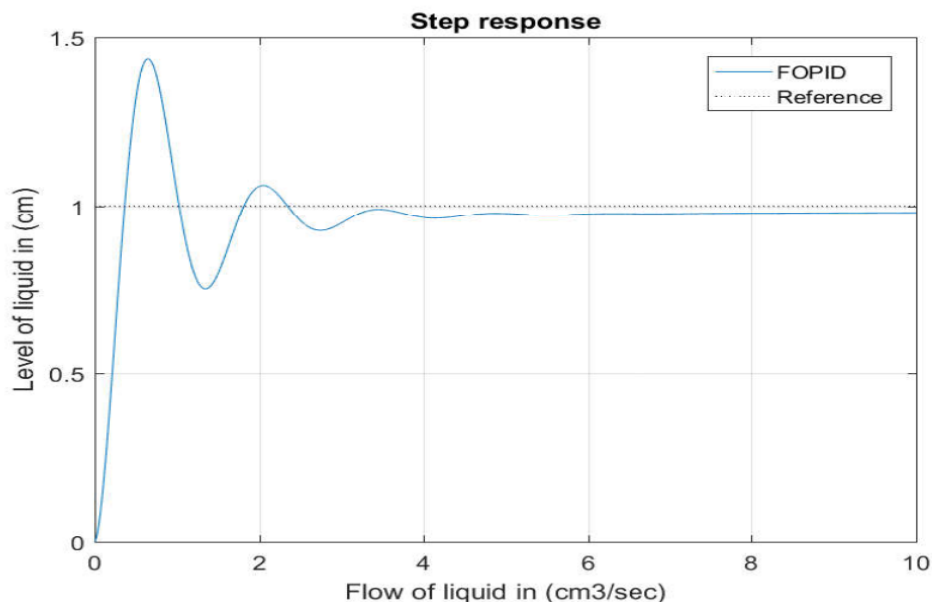


Figure 4.19:-The closed-loop of fractional-order response

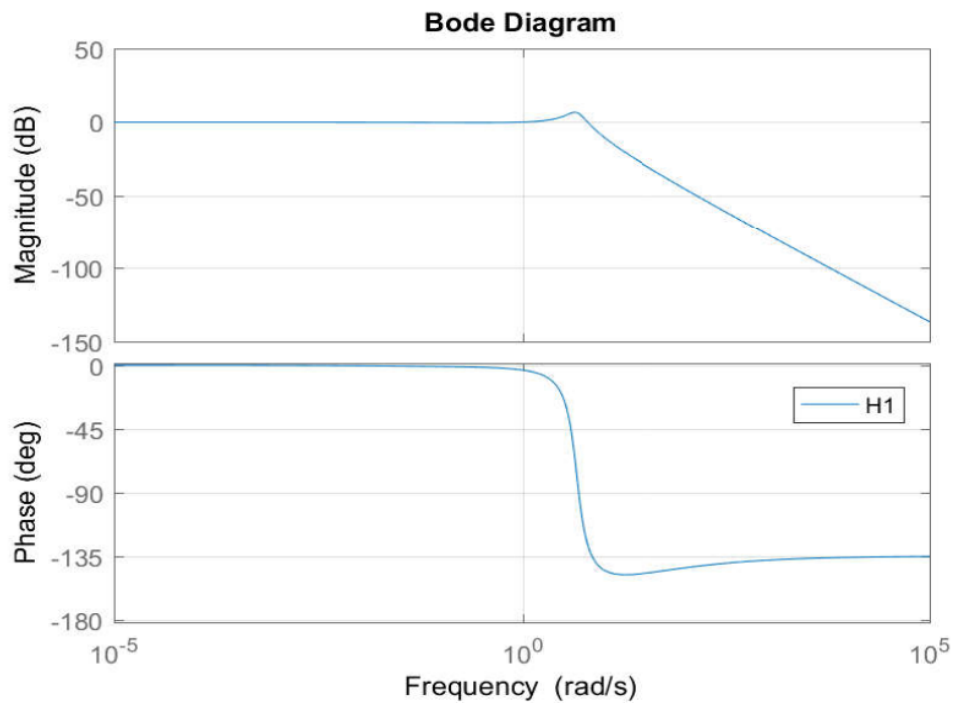


Figure 4.20:- Frequency-Response of fractional- order system  $G(s)$

## CHAPTER FIVE

### SIMULATION RESULTS AND DISCUSSION

#### 5.1.OPEN LOOP PERFORMANCE OF COUPLED TANK (INTERACTING SYSTEM)

The open-loop performance of the interacting is investigated using its MATLAB/Simulink model, the parameters used to carry out the simulation studies are provided in Table 4.6.

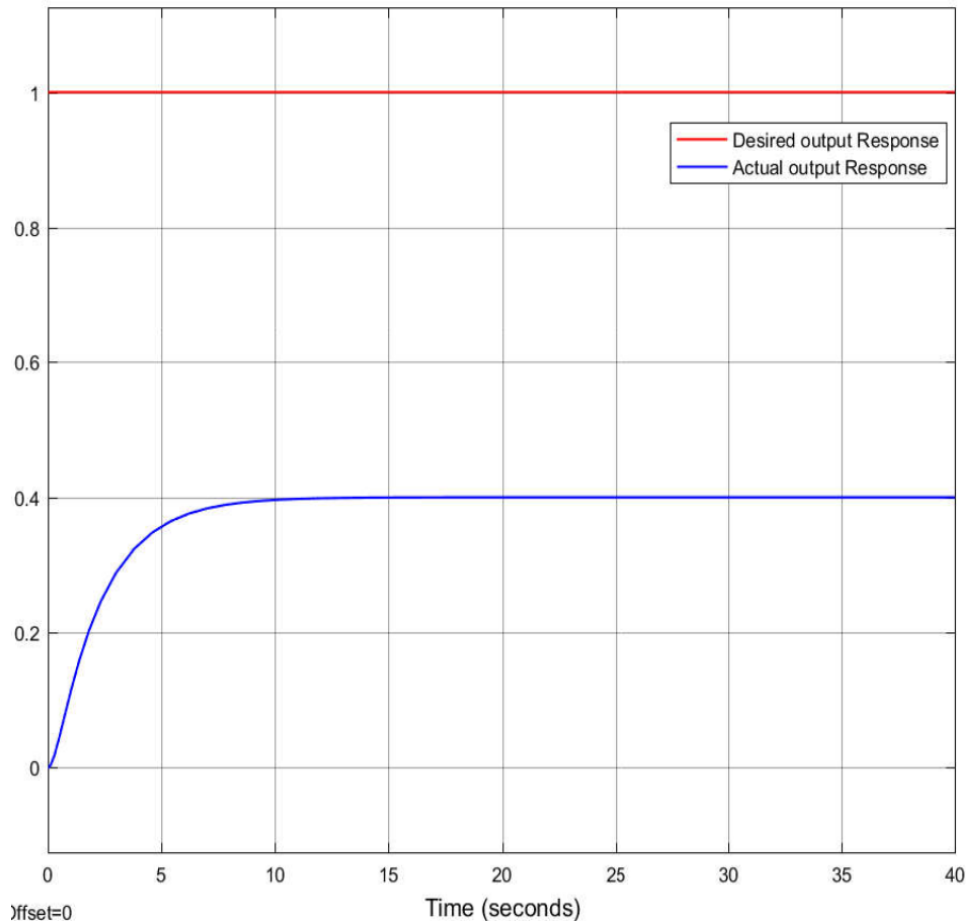


Figure 5.1:- Open-loop output response of interacting two-tank

The output response of the coupled tank (interacting system) obtained using the MATLAB/Simulink is shown in Fig. 5.1. It is observed that the rise time of output interacting system coupled tanks are 4.629 seconds and the settling time is 8.52 seconds.

#### 5.2.PERFORMANCE OF INTERACTING COUPLED TANK WITH PID CONTROLLER

To demonstrate the performance of the interacting coupled tank with a PID controller using in MATLAB/Simulink, the parameter of the level control is given in table 4.6. A

Simulink prototypical of the interacting coupled-tank with the PID controller is given in figure 5.2.

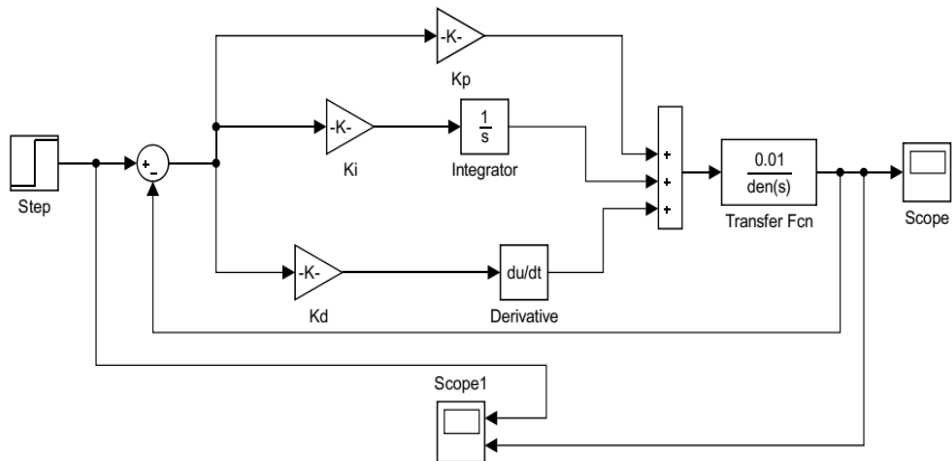


Figure 5.2:- MATLAB/Simulink model of the coupled tank within PID

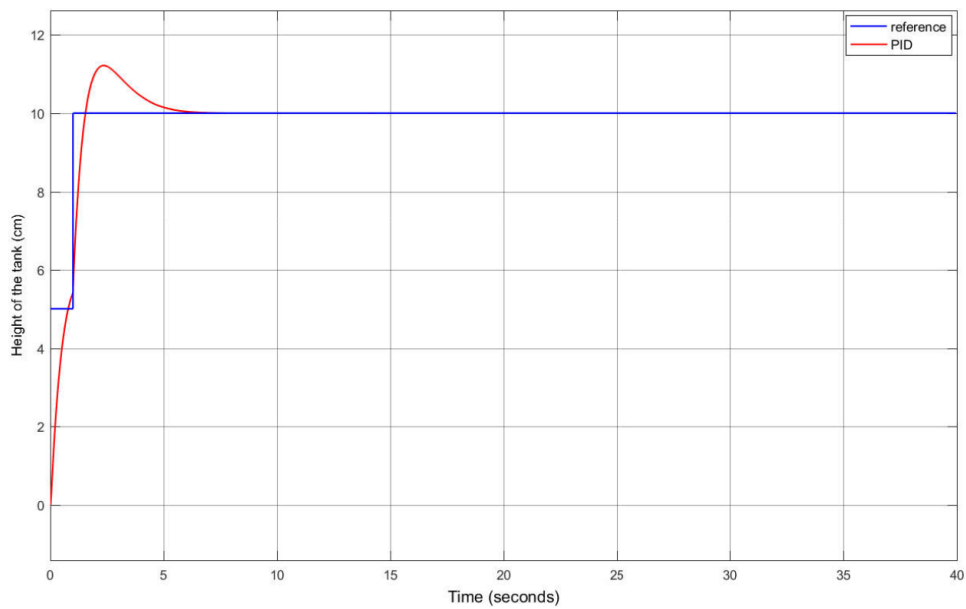


Figure 5.3:-The output response a CT with a PID

AS a result, obtained abovefigure using the PID controls for the coupled interacting tank, the overshoot is 15.117% and rise time 558.939msec.

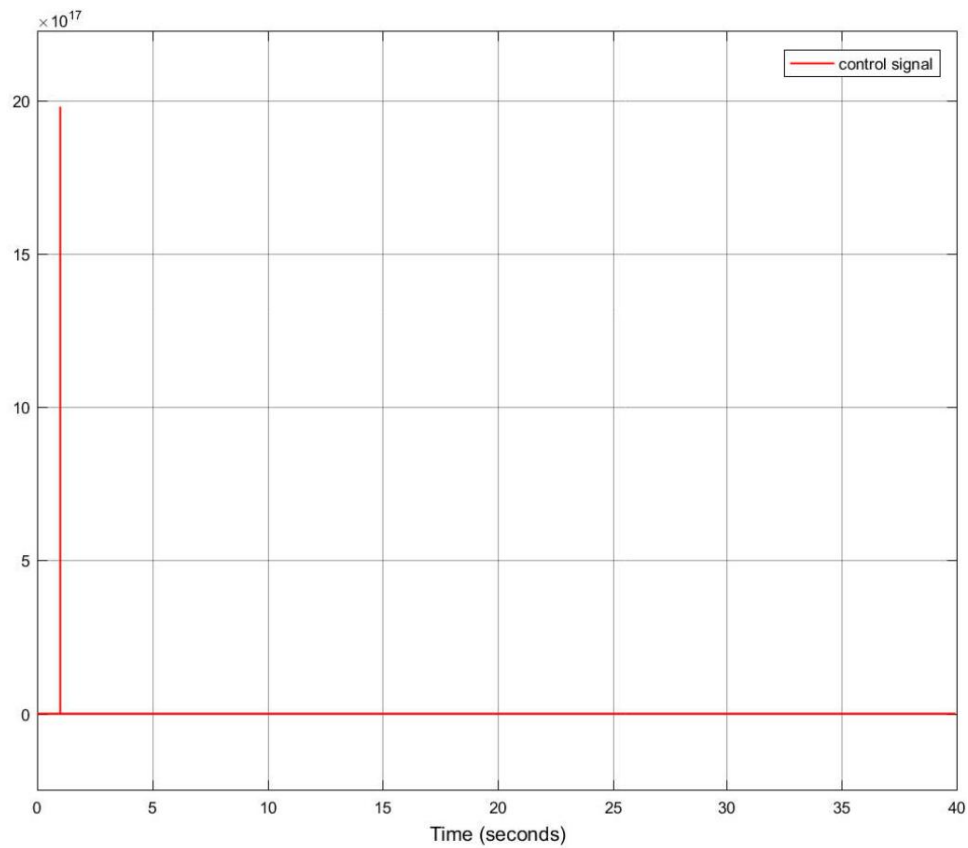


Figure 5.4:- Control signal generated by the PID controller

### 5.3.PERFORMANCE OF TWOCOUPLED TANK WITH FRACTIONAL ORDER PID CONTROLLER

To demonstrate the performance of coupled interacting tank with fractional order PID controller in MATLAB/Simulink, the parameters of the level control are the same as given in table 4.6. The MATLAB/Simulink models of interacting tanks with FOPID are given in below.

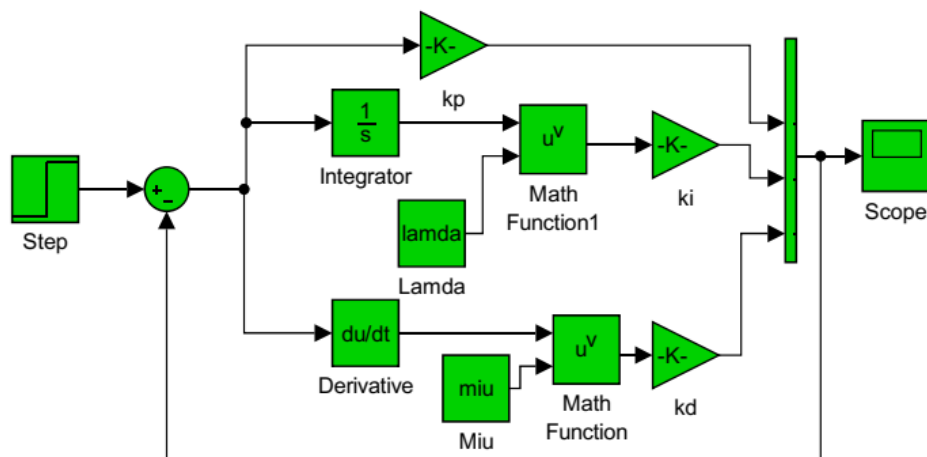


Figure 5.5:- Simulink model of the coupled tank with fractional order PID controller

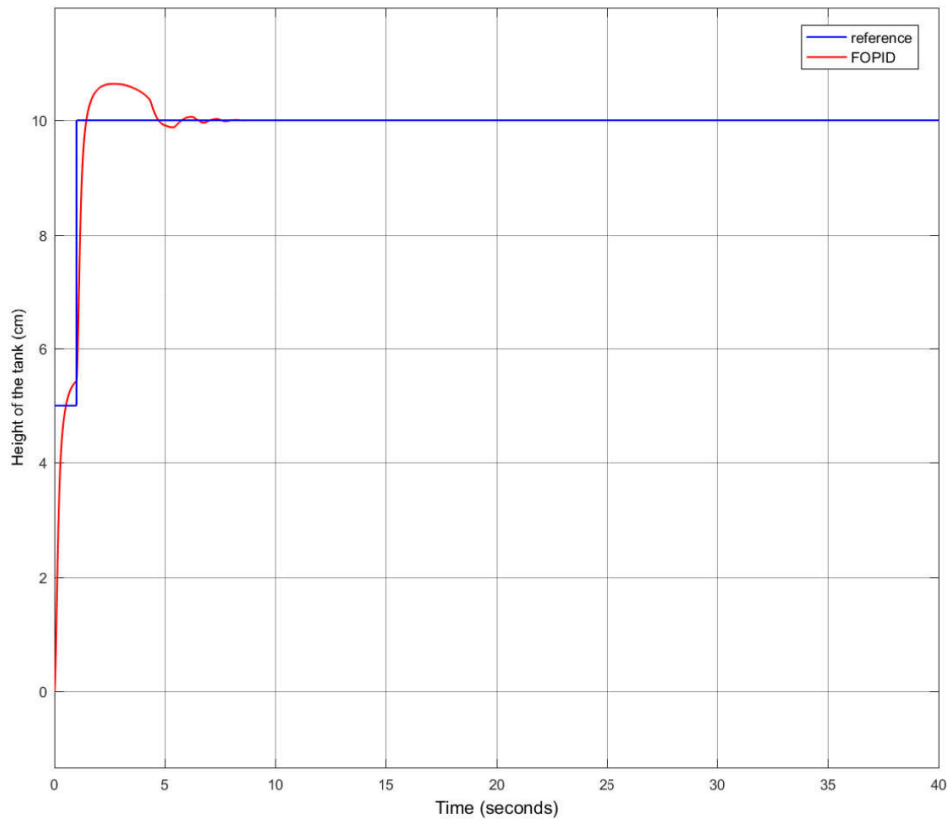


Figure 5.6:- Output of coupled tank with fractional order PID controller

As we observed from figure 5.5 the result obtained in Figure 5.6 using fractional order PID controller for the coupled interacting tank if the parameter value  $\lambda$  and  $\mu$ , in between 0.35 and 0.75 the overshoot is 6.603%, The desired output of the interacting level tank is obtained Rise time 244.703msec.

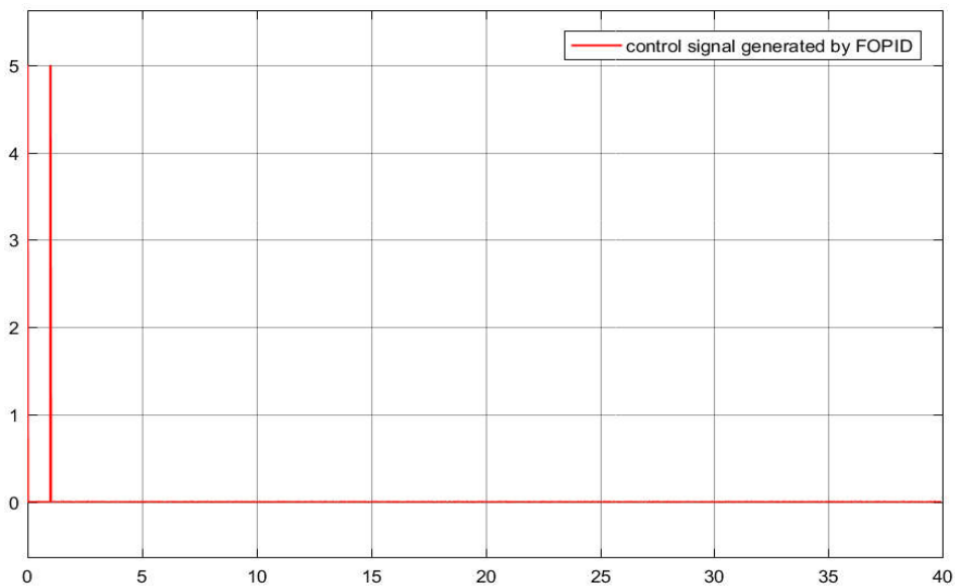


Figure 5.7:- Control signal generated by FOPID controller

#### 5.4.PERFORMANCE OF COUPLED INTERACTING TANK WITH TID CONTROLLER

To demonstrate the performance of coupled interacting tanks with the TID controller in MATLAB/Simulink, the parameters of the level control are the same as given in table 4.6.The MATLAB/Simulink model of interacting tanks with TID control is shown in below,

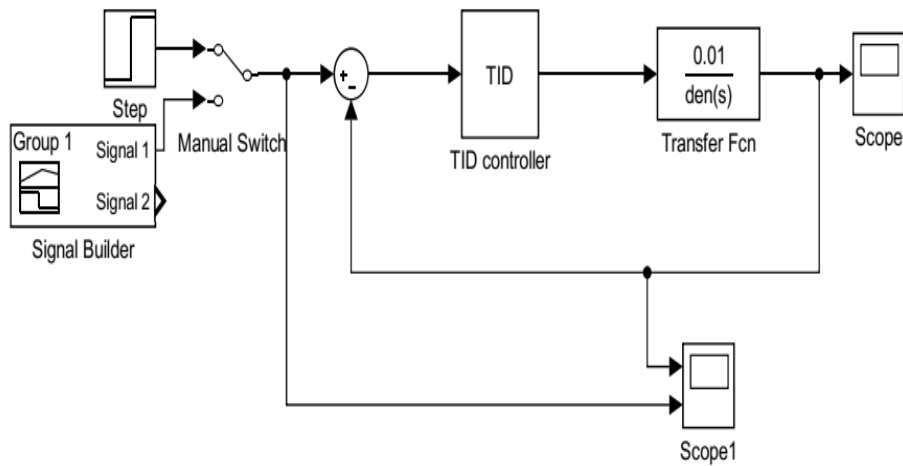


Figure 5.8:- Simulink Model of the CT with TID controller

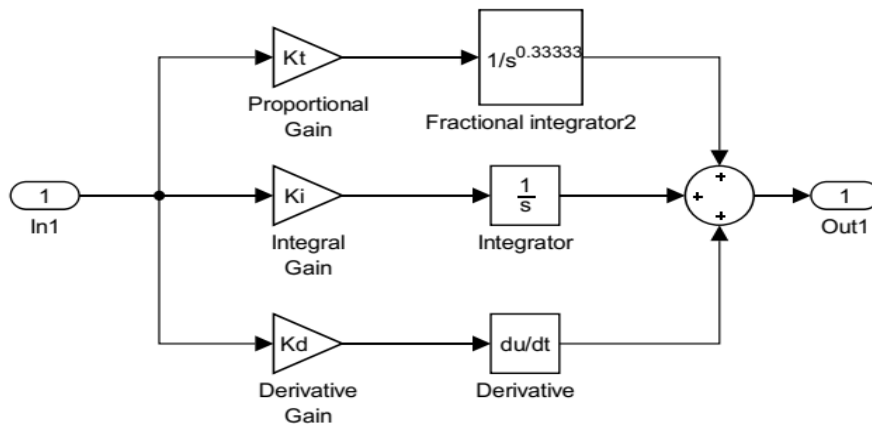


Figure 5.9:-Internal of tilted integral derivative (TID) controller

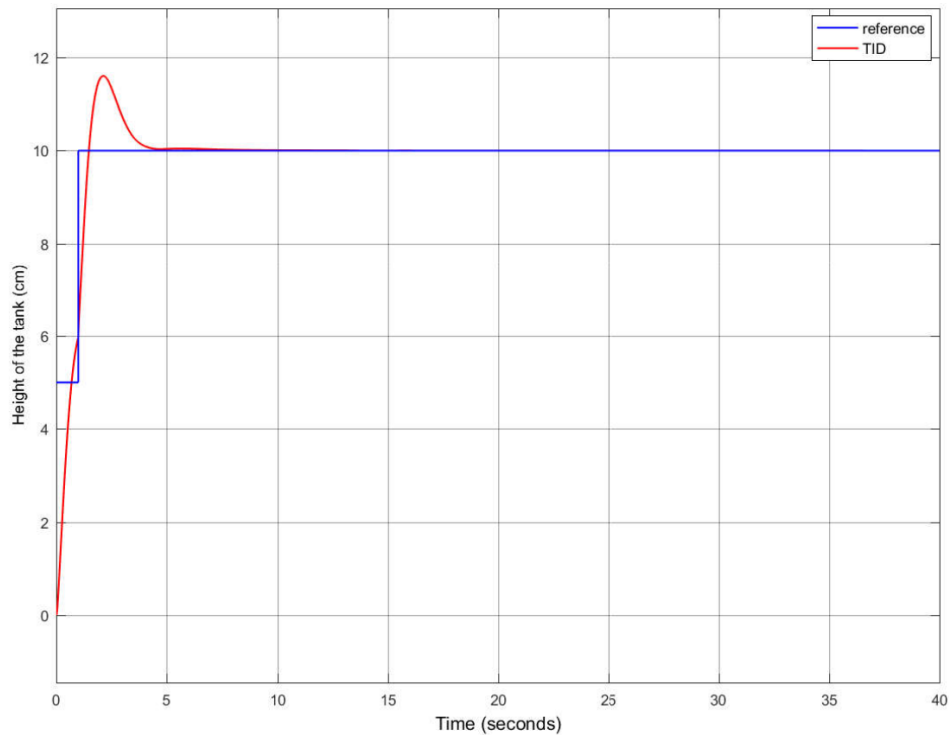


Figure 5.10:- The output response of (CT)with TID controller

From the result obtained in Figure 5.10 using the TID controls for the (CT) the output response of a system below the setpoint when the value of  $n$  is an increase, but if the value of  $n$  equals to five the overshoot is 23.268% the desired output of the interacting level tank is obtained at the Rise time 504.755msec.

### 5.5.PERFORMANCE OF COUPLED INTERACTING TANK WITH THE FRACTIONAL LEAD-LAG COMPENSATOR

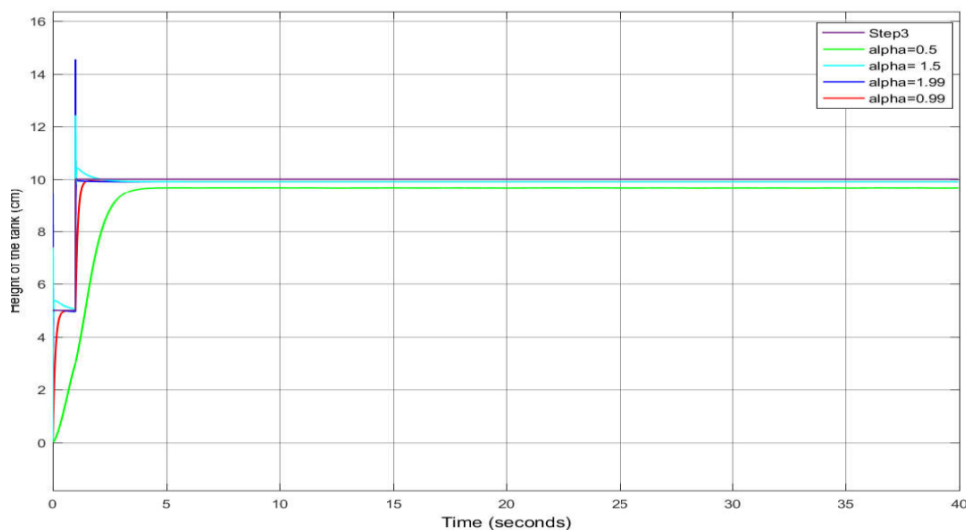


Figure 5.11:- output response of fractional order Lead-Lag controller

As we observed from figure 5.11 fractional-order lead-lag compensator when alpha is between zero and two mean that ( $0 \leq \alpha < 2$ ) if the value of alpha is an increase from 0.5 to 1.99, the system controller response fast according to shown figure 5.11 but not in line with the desired set point

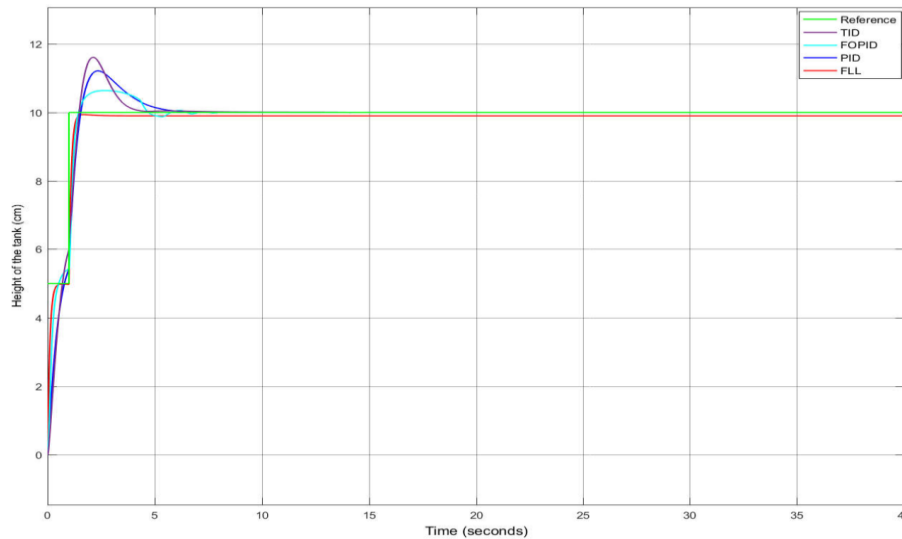


Figure 5.12:- the output response of a system using different controllers

#### 5.6.PERFORMANCE OF COUPLED INTERACTING TANK WITH LQI FOR TRACKING REFERENCE STATE FEEDBACK

To demonstrate the performance of coupled interacting tank with linear quadratic with integral action for tracking reference in MATLAB/Simulink, the parameters of the level control are the same as given in table 4.6. The MATLAB/Simulink model of interacting tanks with the LQI is shown in figure 5.13.

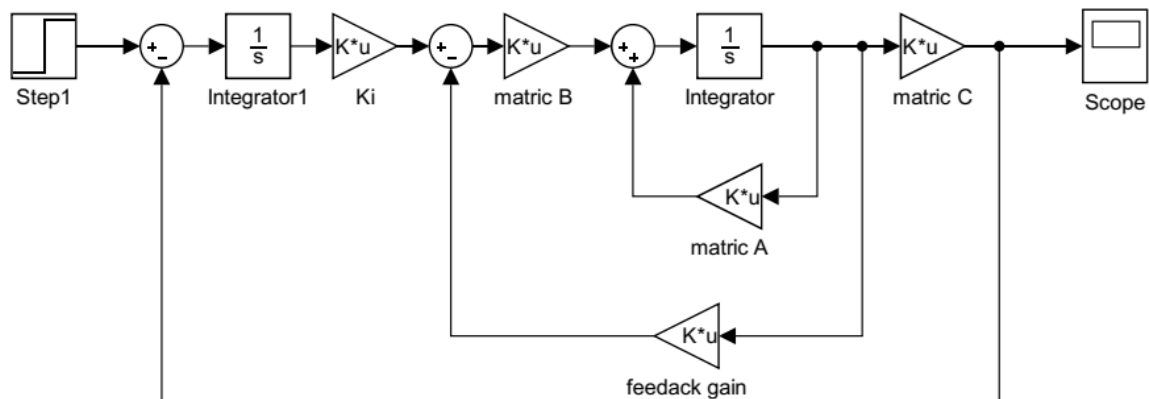


Figure 5.13:- Design coupled tank using LQI controller for tracking Reference



Figure 5.14:- output response of the coupled tank by LQI state feedback

From the result obtained in Figure 5.14 using a LQI feedback the coupled interacting tank, the output of the system overshoot is 7.69%. The desired output of the interacting level tank is obtained at the rise time 81.013msec..

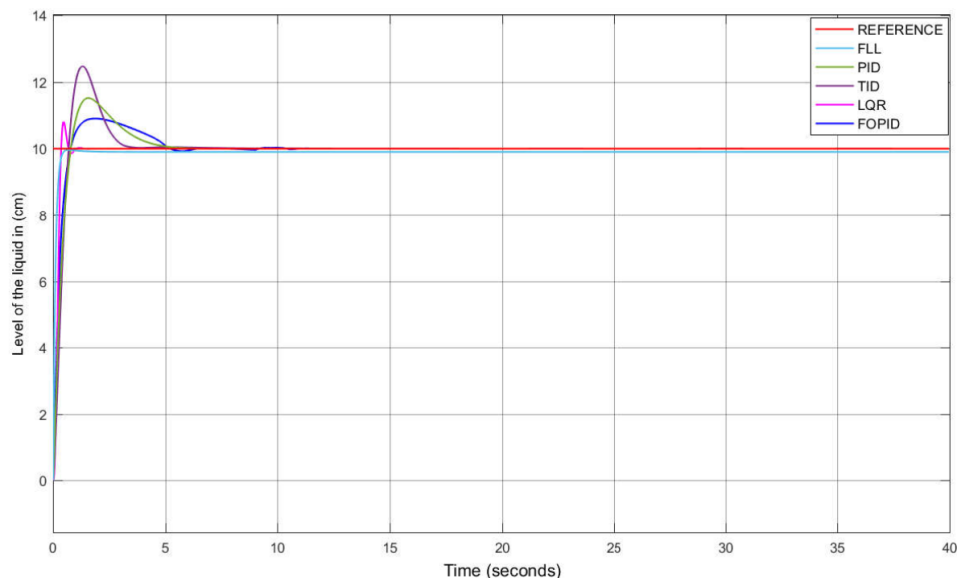


Figure 5.15:- The Over Output Response of Fractional order Controller and LQI Controllers

## CHAPTER SIX

### CONCLUSIONS AND RECOMMENDATION

#### 6.1.CONCLUSIONS

Process control is essential in the industrial process because it guarantees the safety and Optimization is a process. Additionally, process control is a useful tool to satisfy the environmental procedure and product quality necessities. For chemical engineers, process control is widely appropriate in the manufacturing system. The governing the height of the container in an industrial process is the main issue, for that case design the appropriate controller, performance comparison of process-level control in case of the coupled tank, better performance, in this thesis work you have analysis two-controller those are fractional order controller and linear quadratic controller. The FOPID controller is the enlargement of the conventional PID controller based on fractional calculus. For numerous years, in industries, an IOPID control has been common in the function of industrial control. The greatness involves the clarity of design and its best achievement, such as a low percentage of over each and small settling time (In slow industrial processes which are essential. Fractional-order proportional integral derivative-controller have two more adaptable constraints than the PID-controller, &the order of the regulator can be selected randomly. TID Controller is to deliver a better feedback loop controller having the asset of the conventional PID controller, but if and only if a response which is nearer to the academically optimal response, fractional-order lead-lag controller commonly used to stabilize slightly stable systems and The linear-quadratic- regulator is a control scheme that provides the best conceivable performance concerning some given measure of performance. When using LQR with integral action overcome to design a state feedback controller  $K$  and also compute integral action tracking the reference point. In this technique, a feedback gain matrix  $K$  and LQI reference tracking are designed to use for control strength, the amplitude, and the response system.

By applying and linear quadratic regulator with integral action for state feedback control and fractional order controller, Linear quadratic regulator with integral action(LQI) feedback has a better performance compared to the integer proportional integral derivative and fractional order controller's like(FOPID, TID and Fractional order Lead Lag ) other than I observed from the Matlab/Simulink simulation result of table 5.1 the performance

of Linear quadratic regulator with integral feedback control are better achievement compare to the other.

## 6.2.RECOMMENDATION

Although the intellectual of analysis of the performance of the FOPID controller and optimal linear quadratic control for process-level control in interacting two-tank has been validated by Matlab® simulations, it would be advantageous to examine its practical implementation.

Another research issue is to test another optimal control techniques- algorithm than the used prescribed convergence algorithm. Analyze and evaluate the performance of optimal control techniques designed using different algorithms could lead to new insight.

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## APPENDIX

### Appendix A

% to create the transfer function of coupled tank can be used

$$A_1 = 250\text{cm}^2; A_2 = 250\text{cm}^2; R_1 = 0.01 \frac{\text{sec}}{\text{cm}^2}; R_2 = 0.01\text{sec./cm}^2$$

$$G(s) = \frac{0.0016}{s^2 + 1.2s + 0.16}$$

The output response of open loop system can be

```
>>num=0.0016;
```

```
>>dem= [1 1.2 0.16];
```

```
G(s)=tf (num,den)
```

```
Step (G(s))
```

State space form of the system in controllable canonical form

$$A = \begin{bmatrix} 0 & 1 & 0 \\ -0.16 & -1.2 & 0 \\ -1 & 0 & 0 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 0 \\ 0.0016 \end{bmatrix}, C = [1 \ 0 \ 0], D=0;$$

The value of relative dominance  $\alpha = 1$ , damping ratio of closed loop  $\xi^{cl} = 0.95$  and natural frequency of closed loop  $\omega^{cl} = \frac{2\text{rad}}{\text{sec}}$ .

And also natural frequency, damping ratio of the open loop system a respectively,  $\omega = 0.4$ ,  $\xi = 0.15$ .

By using the formal on page 27, the gain of PID calculated

$$k_p = 6960, k_i = 4750, k_k = 2812.5$$

Linear quadratic regulator design depending on the following parameter value

$$R = 0.04; Q = \begin{bmatrix} 25 * 10^4 - 6 & 0 & 0 \\ 0 & 75 * 10^4 - 3 & 0 \\ 0 & 0 & 20 * 10^9 \end{bmatrix}$$

$$K = lqr(A_{new}, B_{new}, B, q, r)$$

Linear quadratic regulator with integral action for state feedback gain  $k$ .

$$k = 10^5([1.3574 \ 0.123 \ -7.0711])$$

For a figure page 55 of coupled tank controlled with fractional order PID controller when  $\mu = \lambda = 0.5$

```
>>loadsets
```

All test sets loaded successfully.

```
>>kp= 6960;
```

```
>>Ki=4750;
```

```
>> kd=2812.5;
```

```
>> s=fotf('s')
```

Fractional-order transfer function:

s

```
>> num=0.0016;
```

```
>> den=[1 1.2 0.16];
```

```
>> G(s)=tf(num,den)
```

G(s) =

0.0016

-----

$s^2 + 1.2 s + 0.16$

Continuous-time transfer function.

```
>> H(s)=kp*s+ki*s^-0.5+kd*s^0.5
```

Fractional-order transfer function:

$6960s^{1.5}+2812.5s+4750$

-----

$s^{0.5}$

```
>> F(s)=H(s)*G(s)
```

Fractional-order transfer function:

$11.136s^{1.5}+4.5s+7.6$

-----

$s^{2.5}+1.2s^{1.5}+0.16s^{0.5}$

```
>> Y(s)=feedback(F(s),1)
```

Fractional-order transfer function:

$11.136s^{1.5}+4.5s+7.6$

-----

$$s^{2.5} + 12.336s^{1.5} + 4.5s + 0.16s^{0.5} + 7.6$$

>>step(Y(s))

For reduced the overshoot, settling time, rise time response take value of  $\lambda = 0.35$  and  $\mu = 0.75$ . for TID controller the value of  $n=5$ ; and the value of alpha for fractional order lead lag controller used in between zero and two.